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(54) **CROSSBOW WITH A CRANK COCKING AND
RELEASE MECHANISM**

(71) Applicant: **Bahram Khoshnood**, Cumming, GA
(US)

(72) Inventor: **Bahram Khoshnood**, Cumming, GA
(US)

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F41B 5/14 (2006.01)

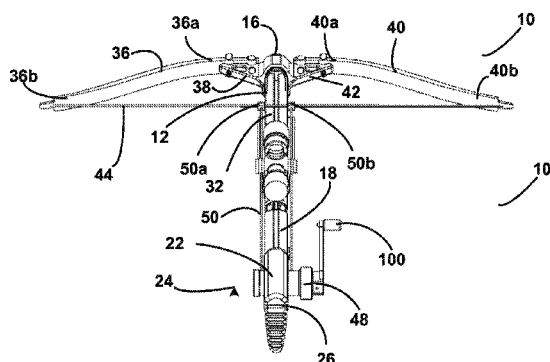
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(2013.01); **F41B 5/123** (2013.01)

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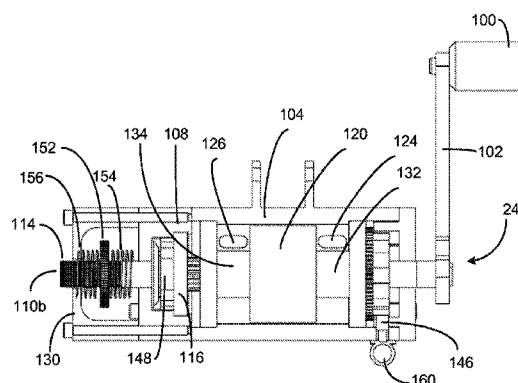
Primary Examiner — Alexander Niconovich

(74) *Attorney, Agent, or Firm* — Brient Globerman, LLC

(57) **ABSTRACT**

A crossbow bowstring drawing mechanism comprises (1) a generally cylindrical housing having a first end and a second end; (2) a shaft rotatably mounted in the housing has a first end that at least partially extends through the housing first end; (3) at least one rope spool received on, and rotationally fixed to, the shaft intermediate the shaft first and second ends; (4) a handle operatively coupled to the shaft; and (5) a clutch mechanism received on the shaft. At least one of the rope spools is configured to attach to a first and a second end of a rope that is configured to be releasably attached to a bowstring. The clutch mechanism is configured to at least temporarily rotationally fix the shaft to the housing to prevent the shaft from rotating in a first direction and an opposite second direction until a force is exerted on the handle to overcome the frictional forces exerted by the clutch mechanism.

20 Claims, 19 Drawing Sheets



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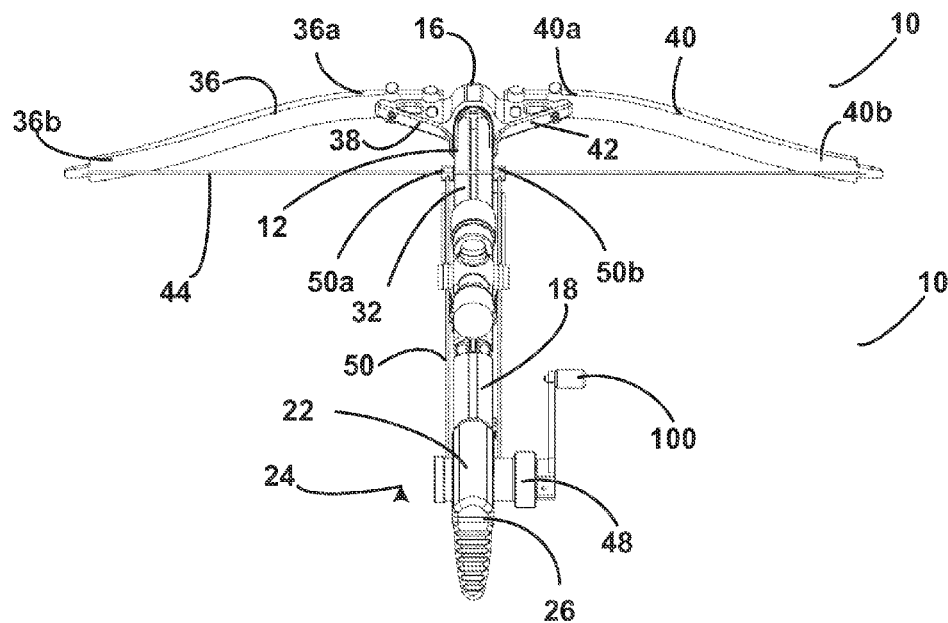


FIG. 1

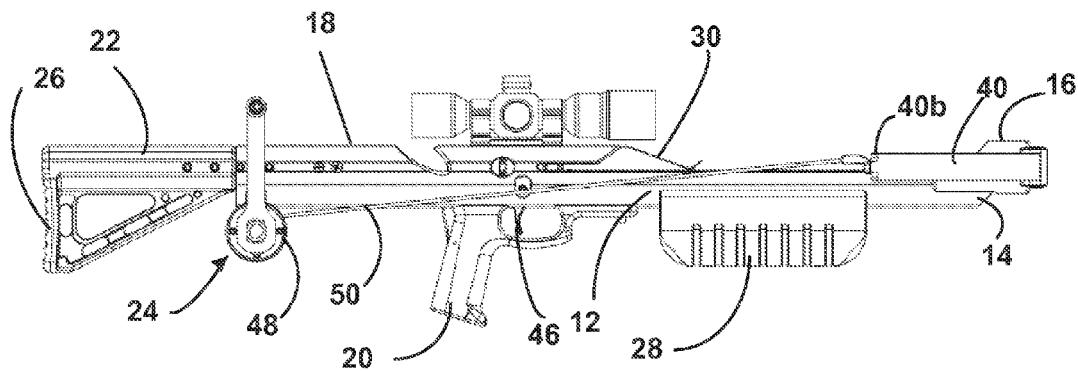
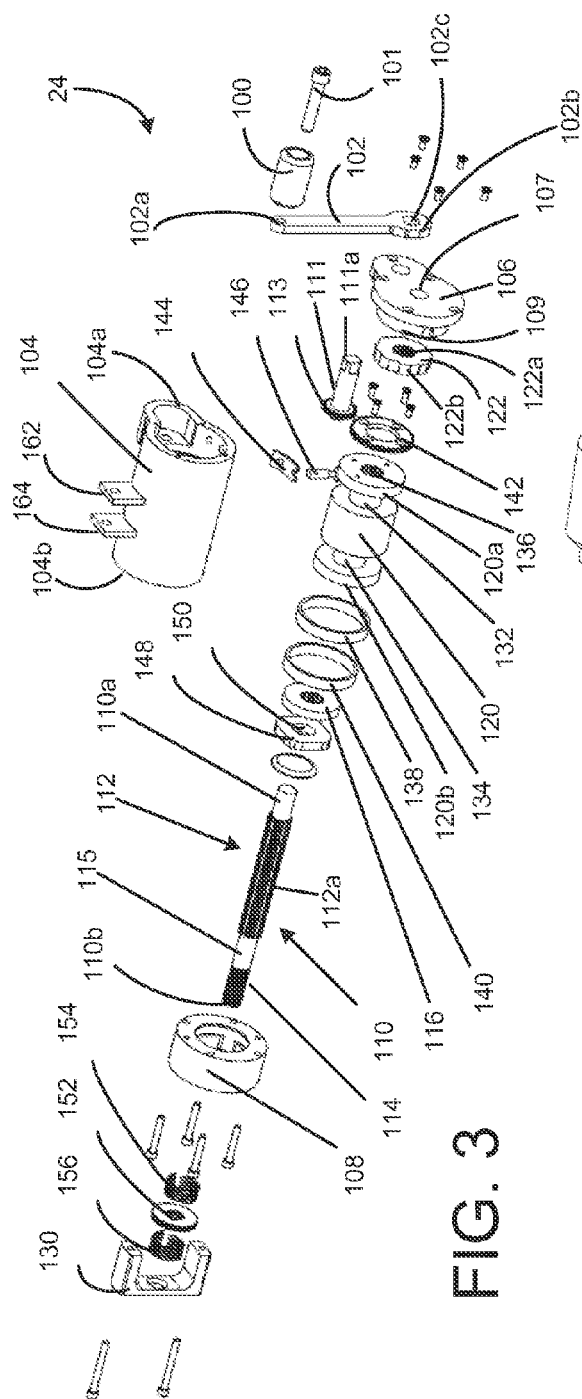
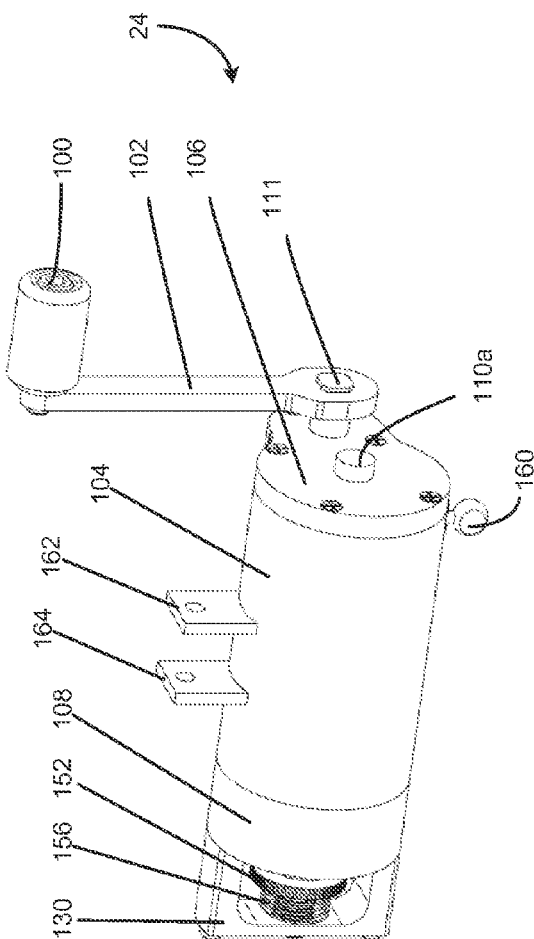


FIG. 2



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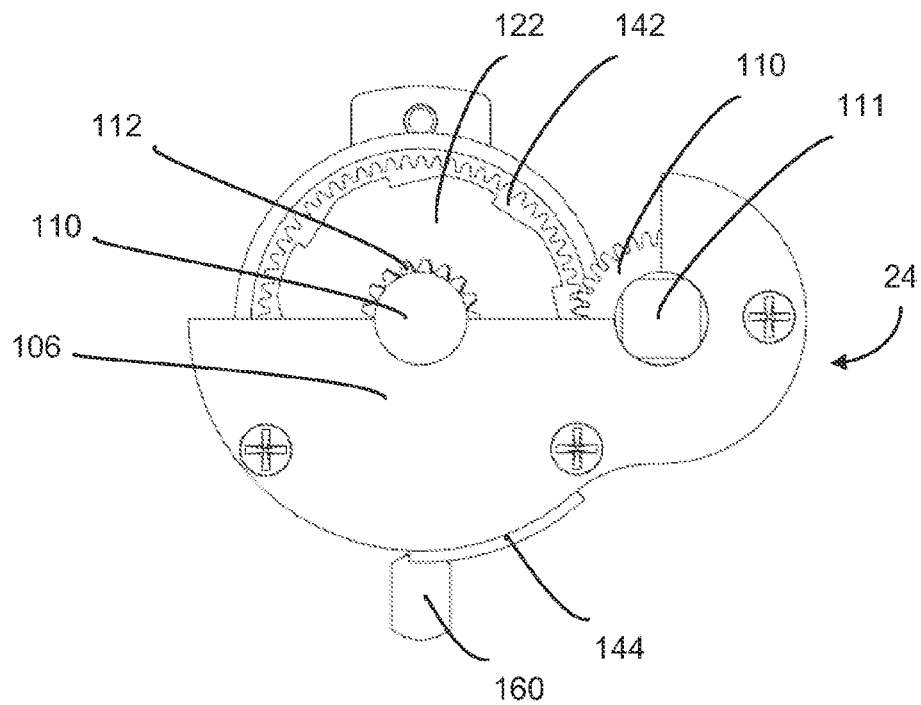


FIG. 5

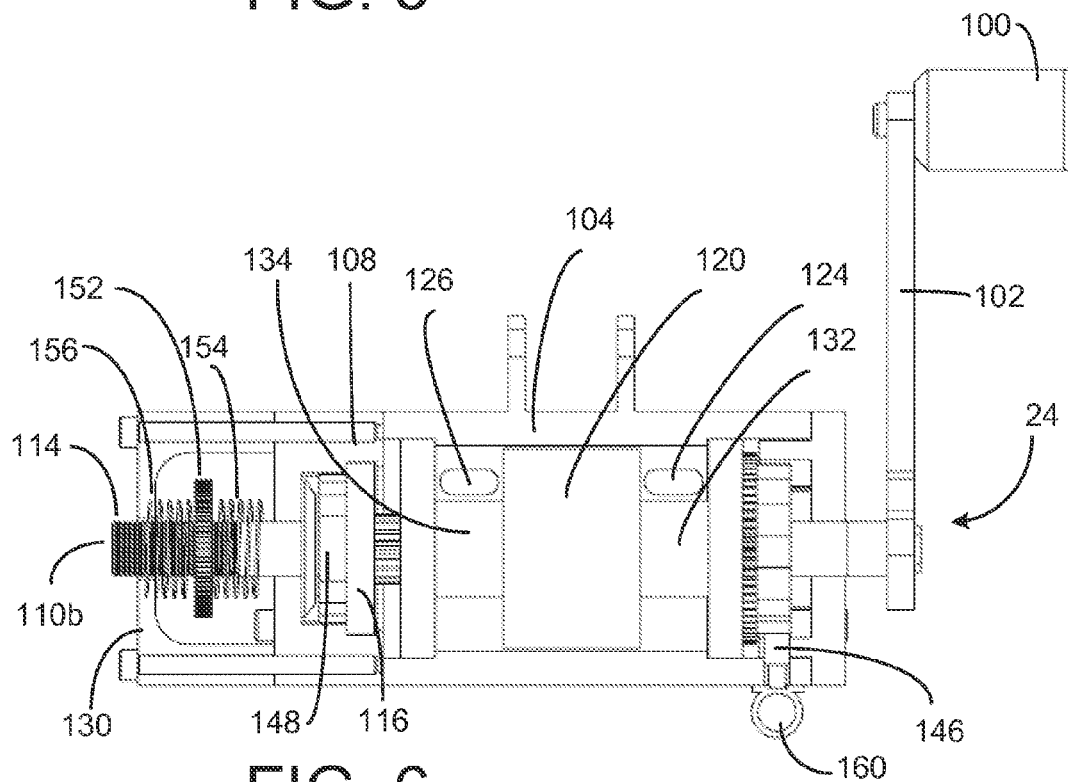


FIG. 6

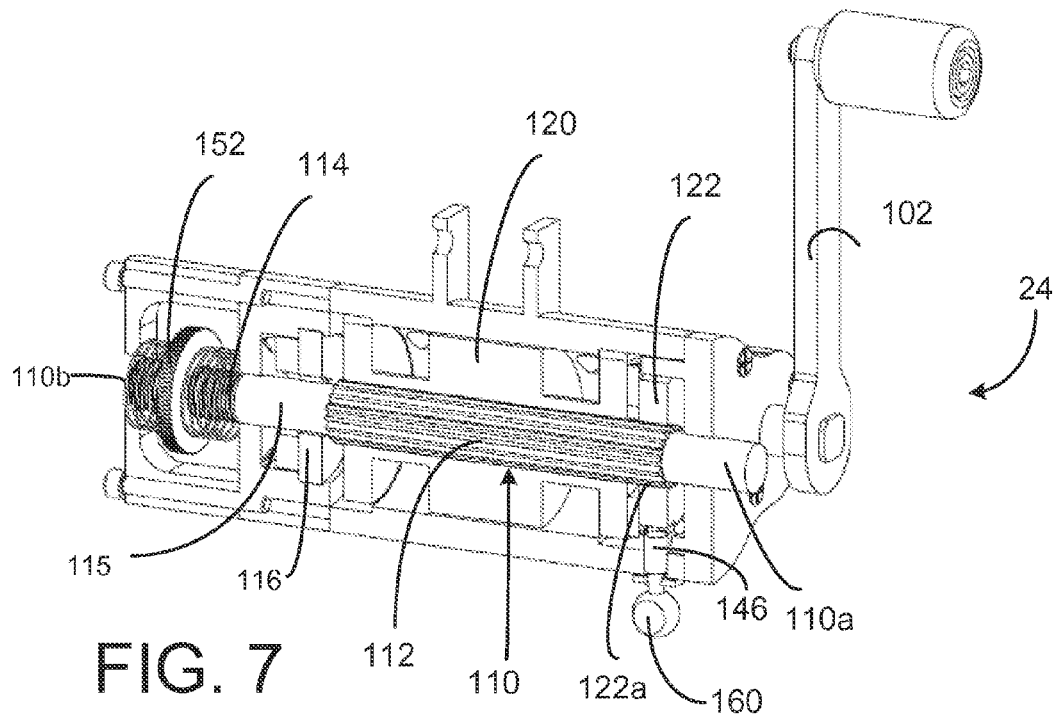


FIG. 7

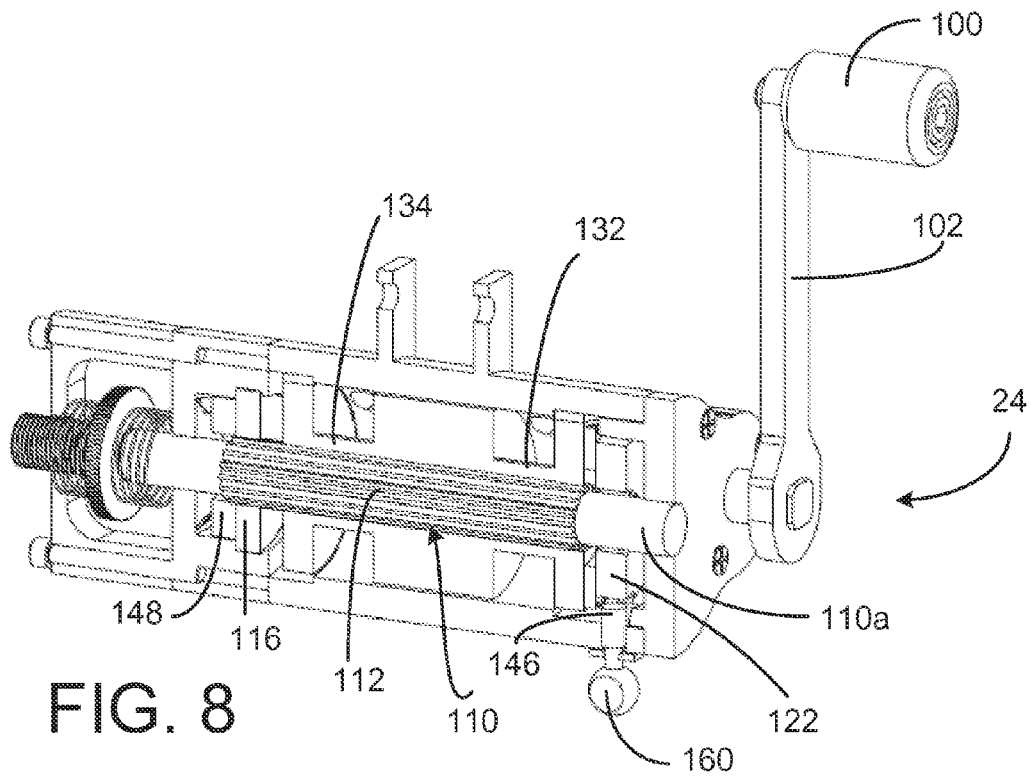


FIG. 8

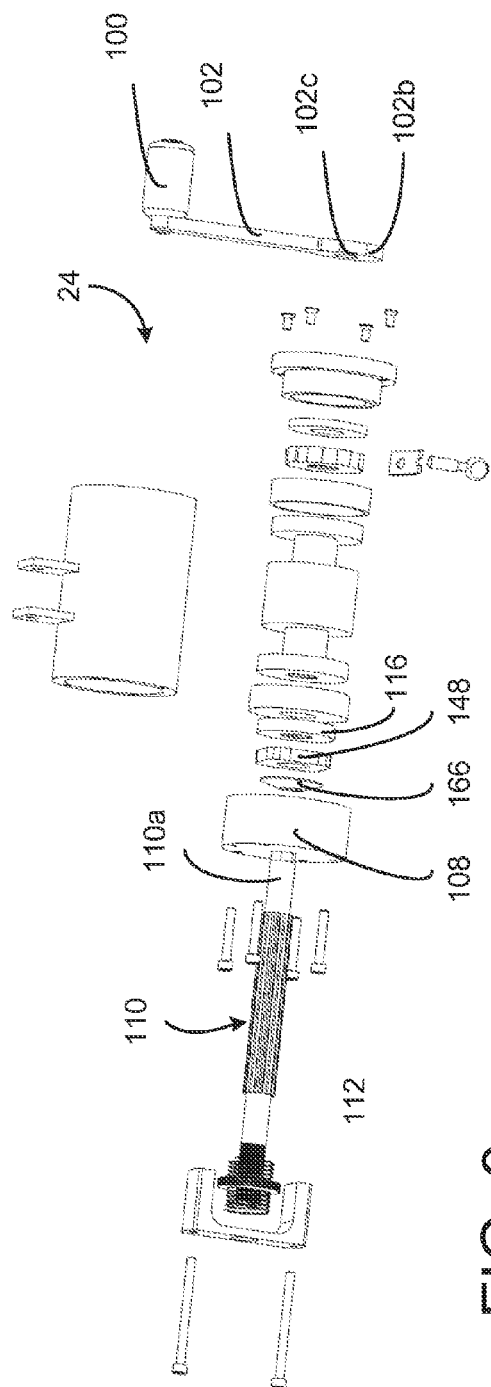


FIG. 9

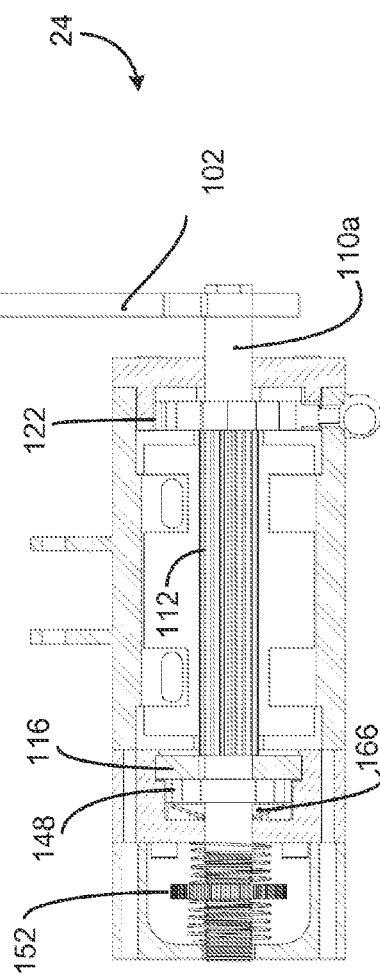
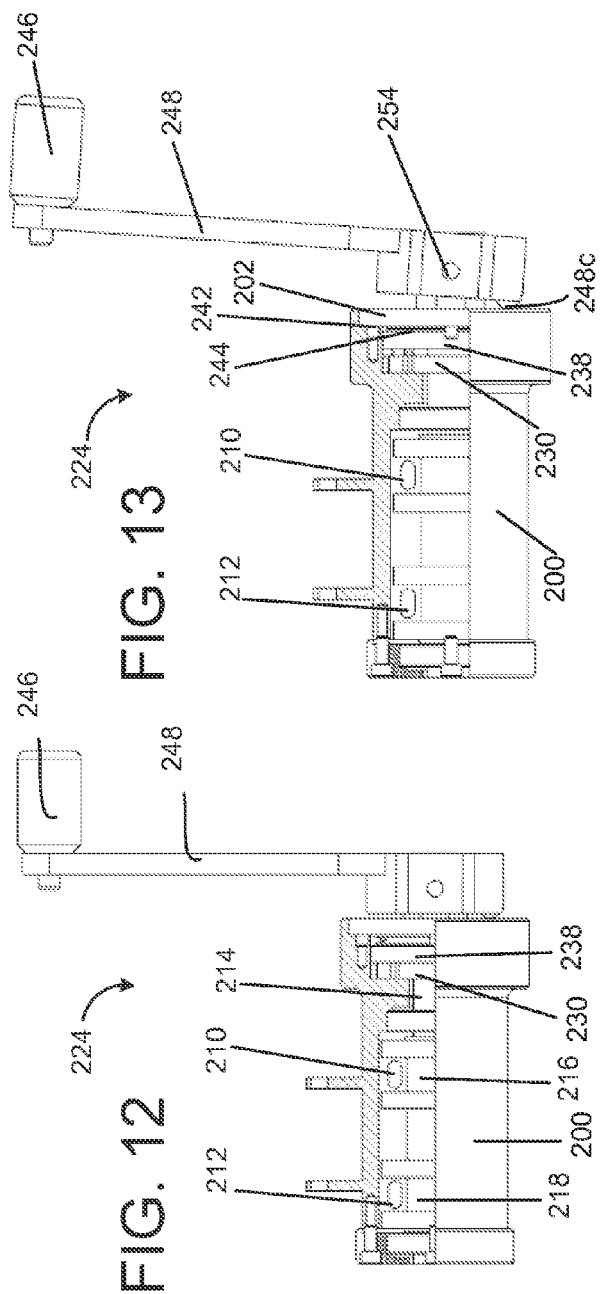
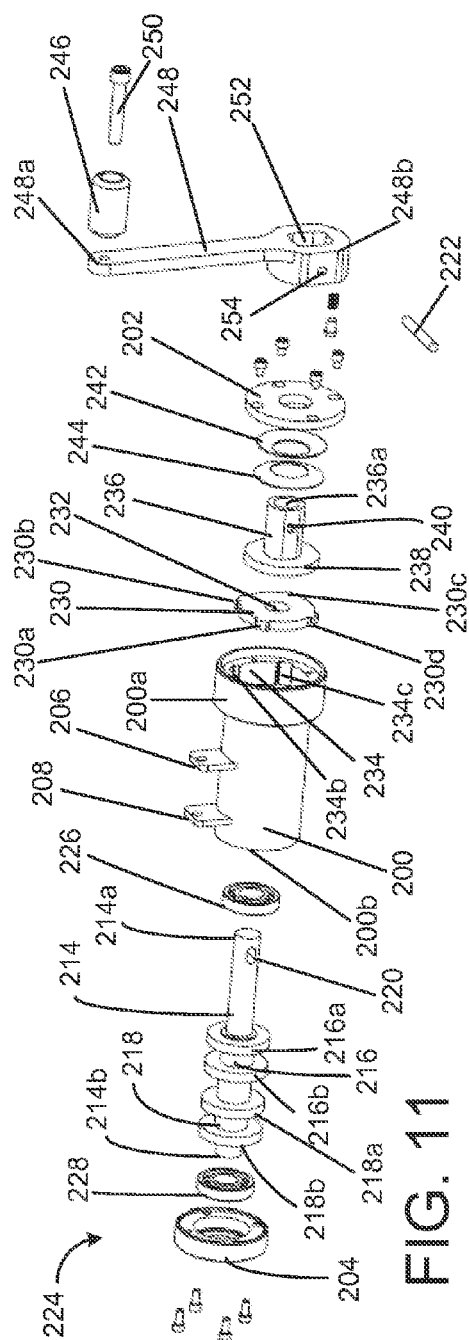


FIG. 10



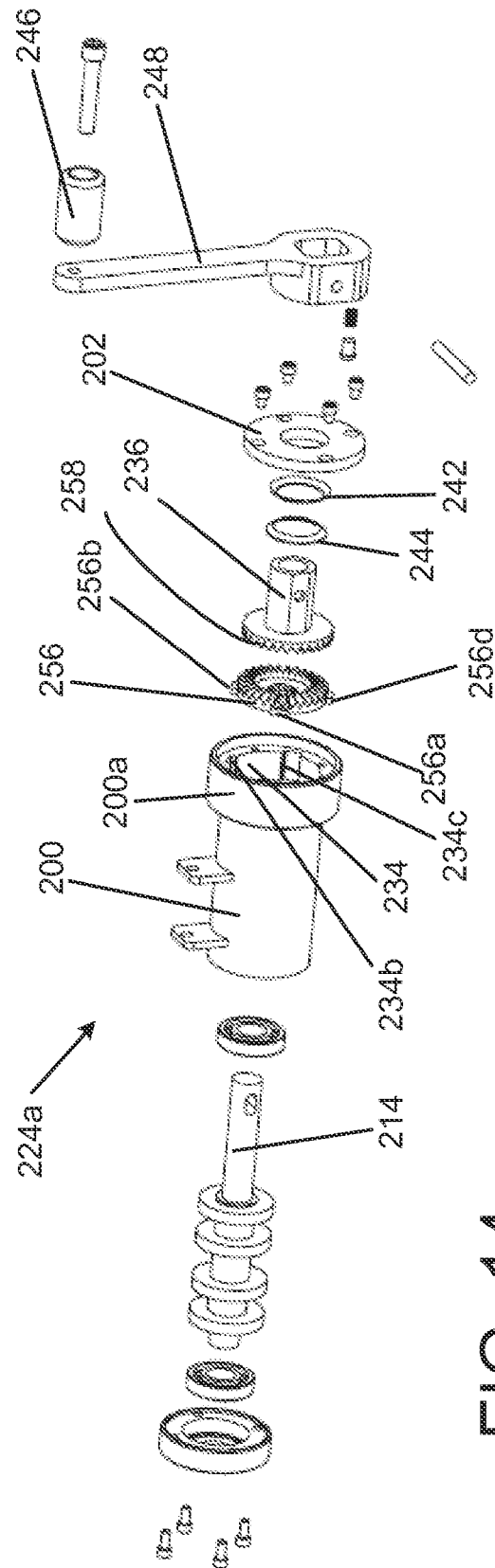
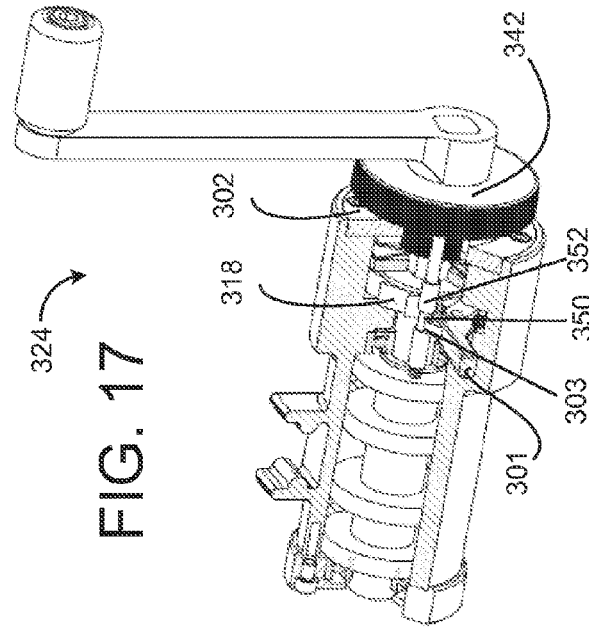
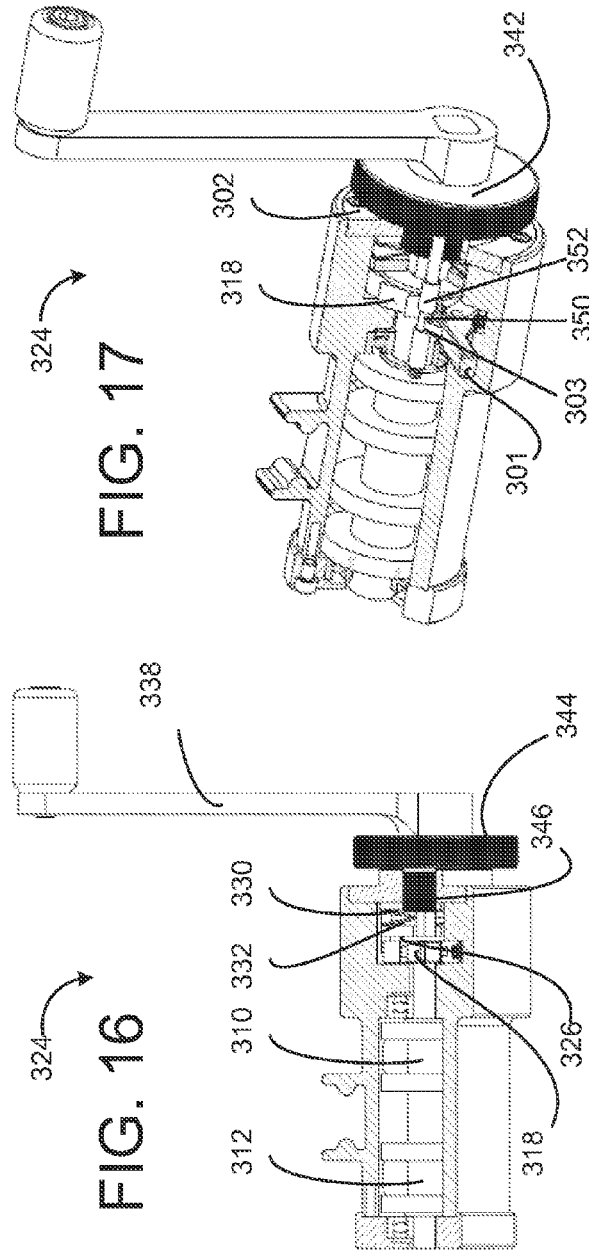
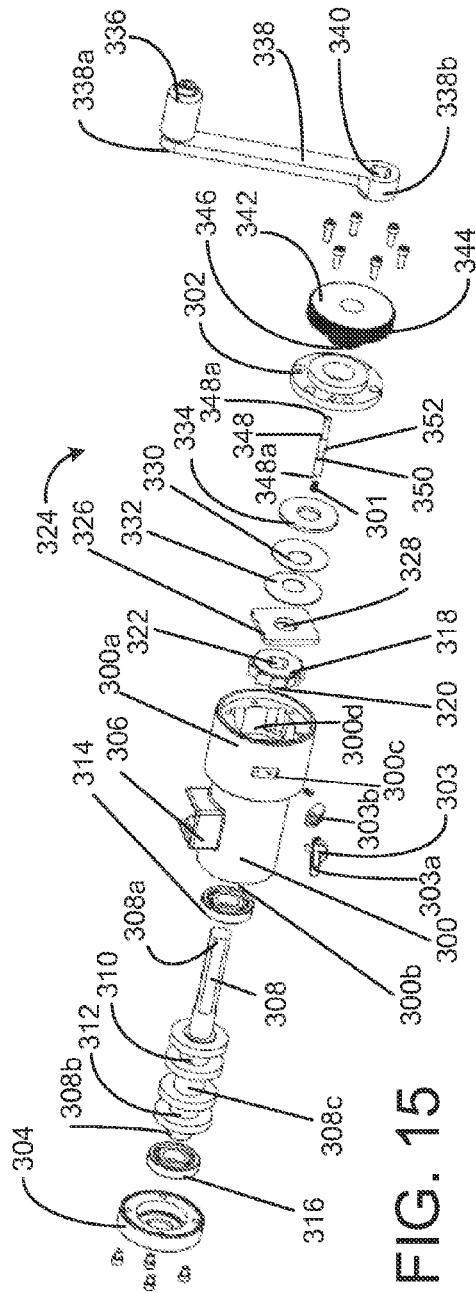
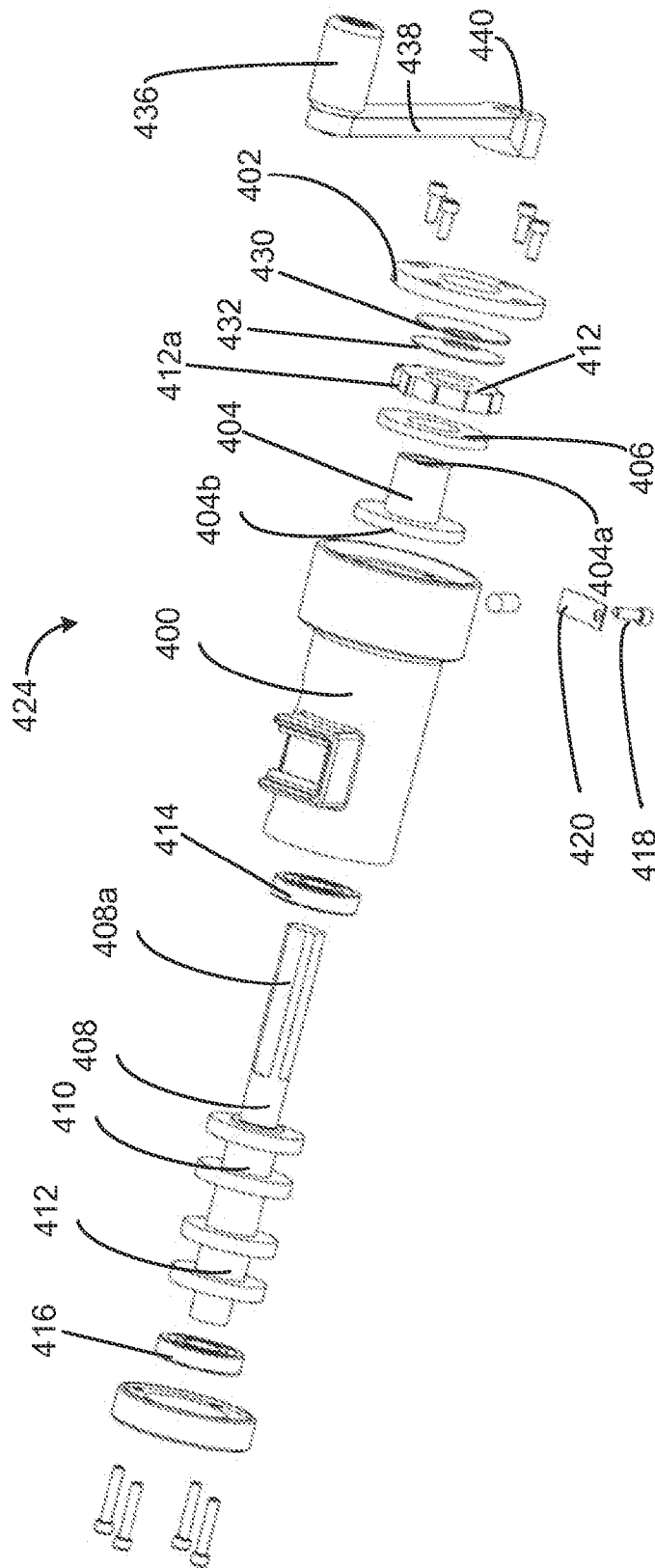



FIG. 14







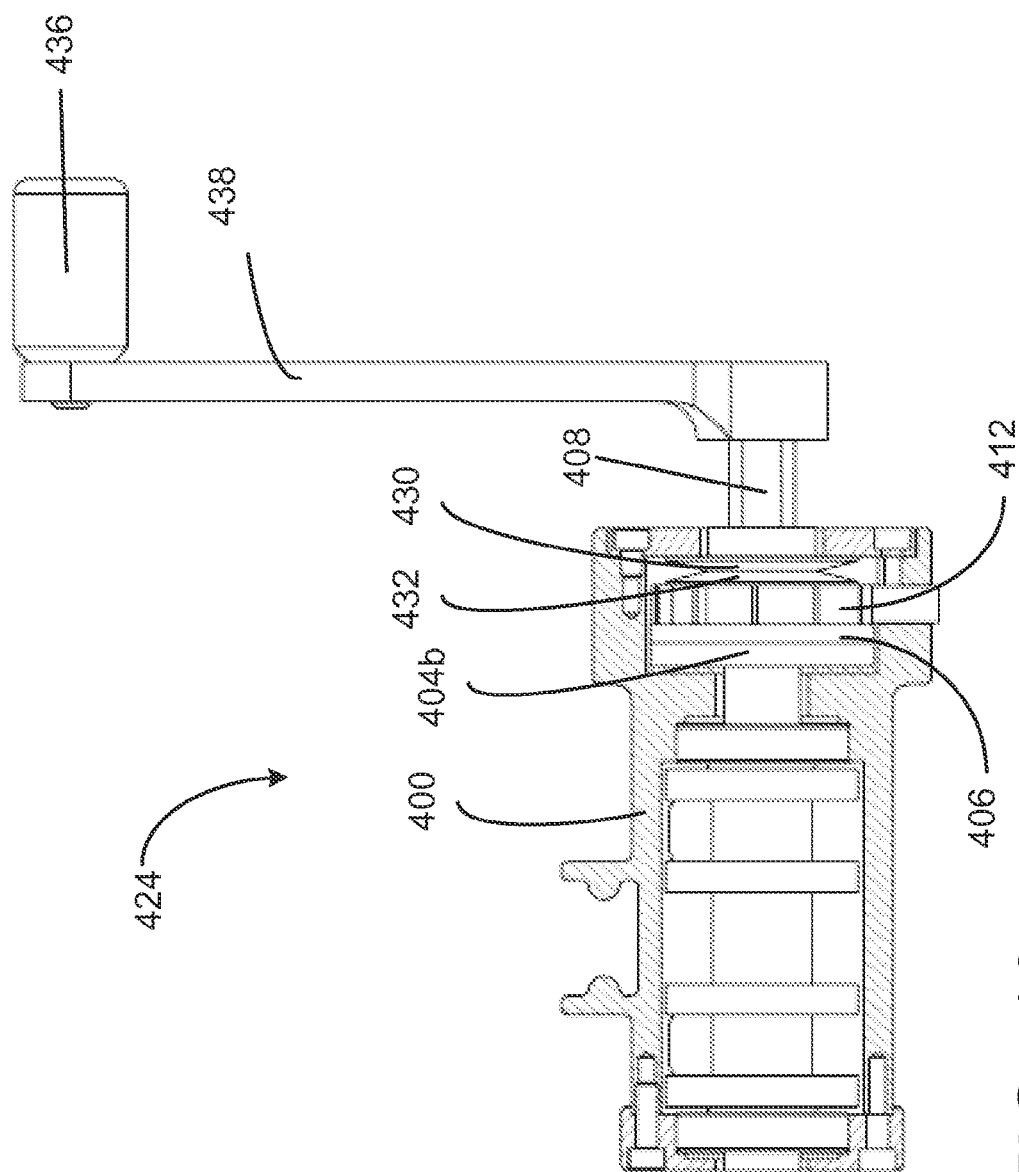
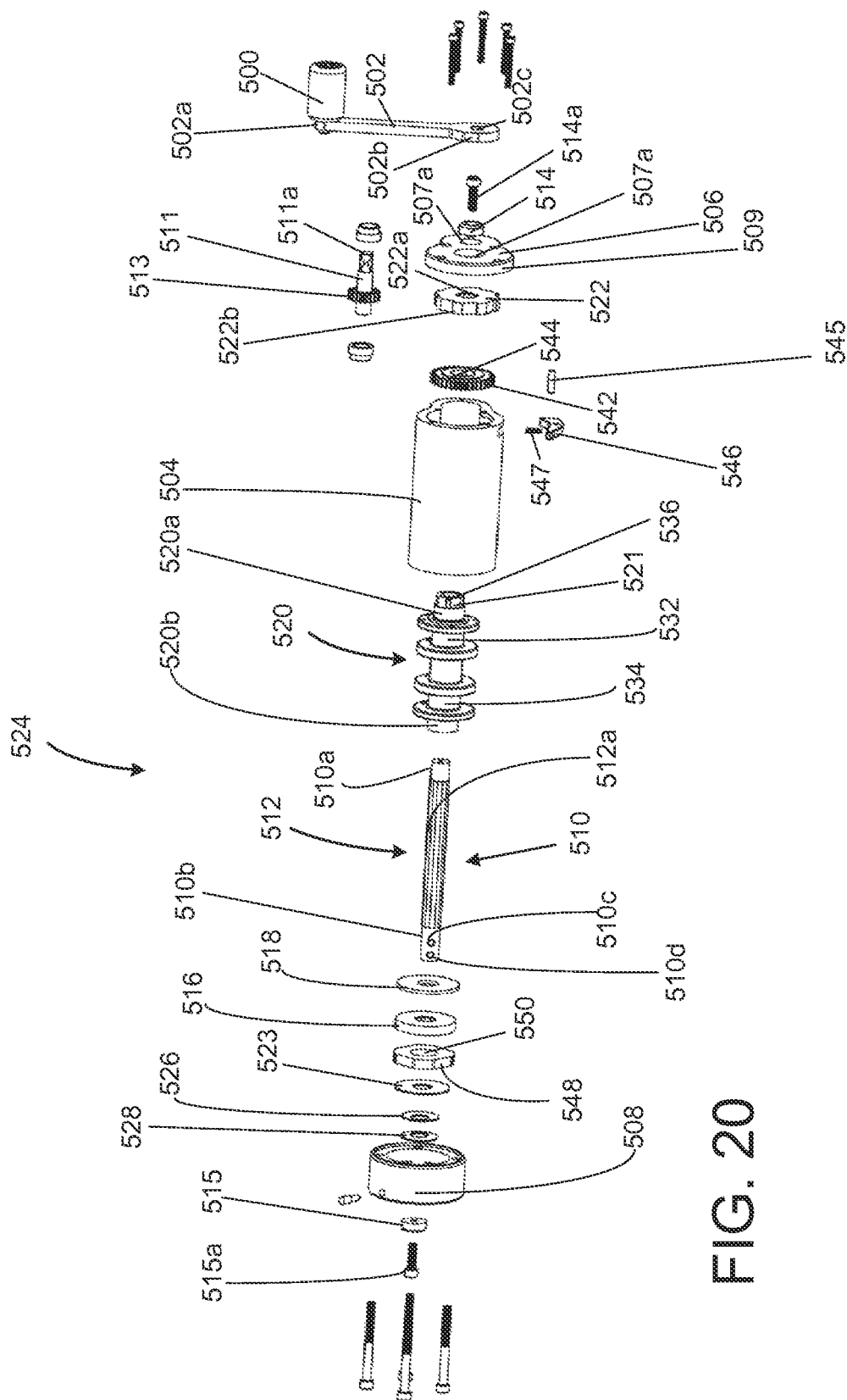


FIG. 19



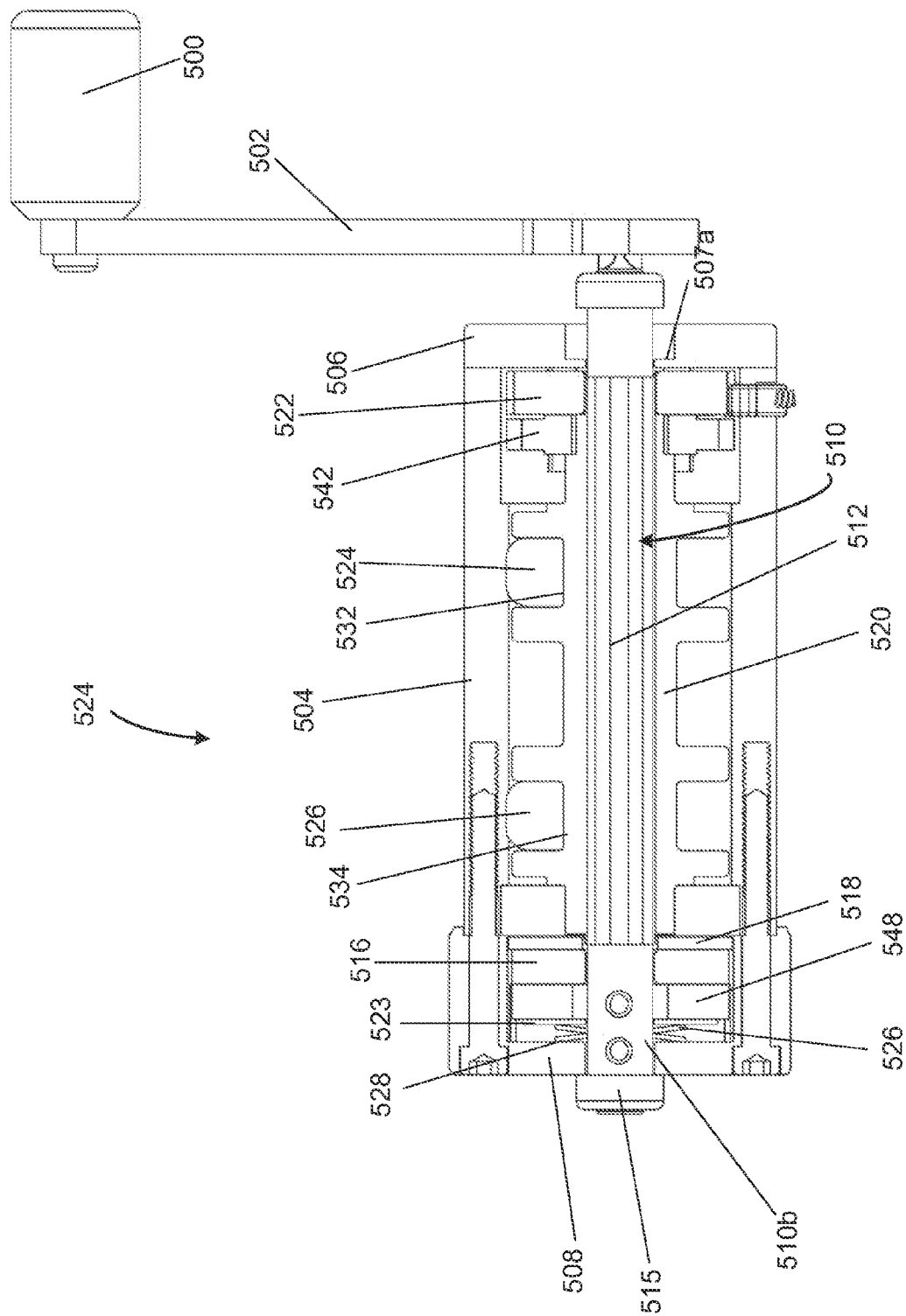


FIG. 21

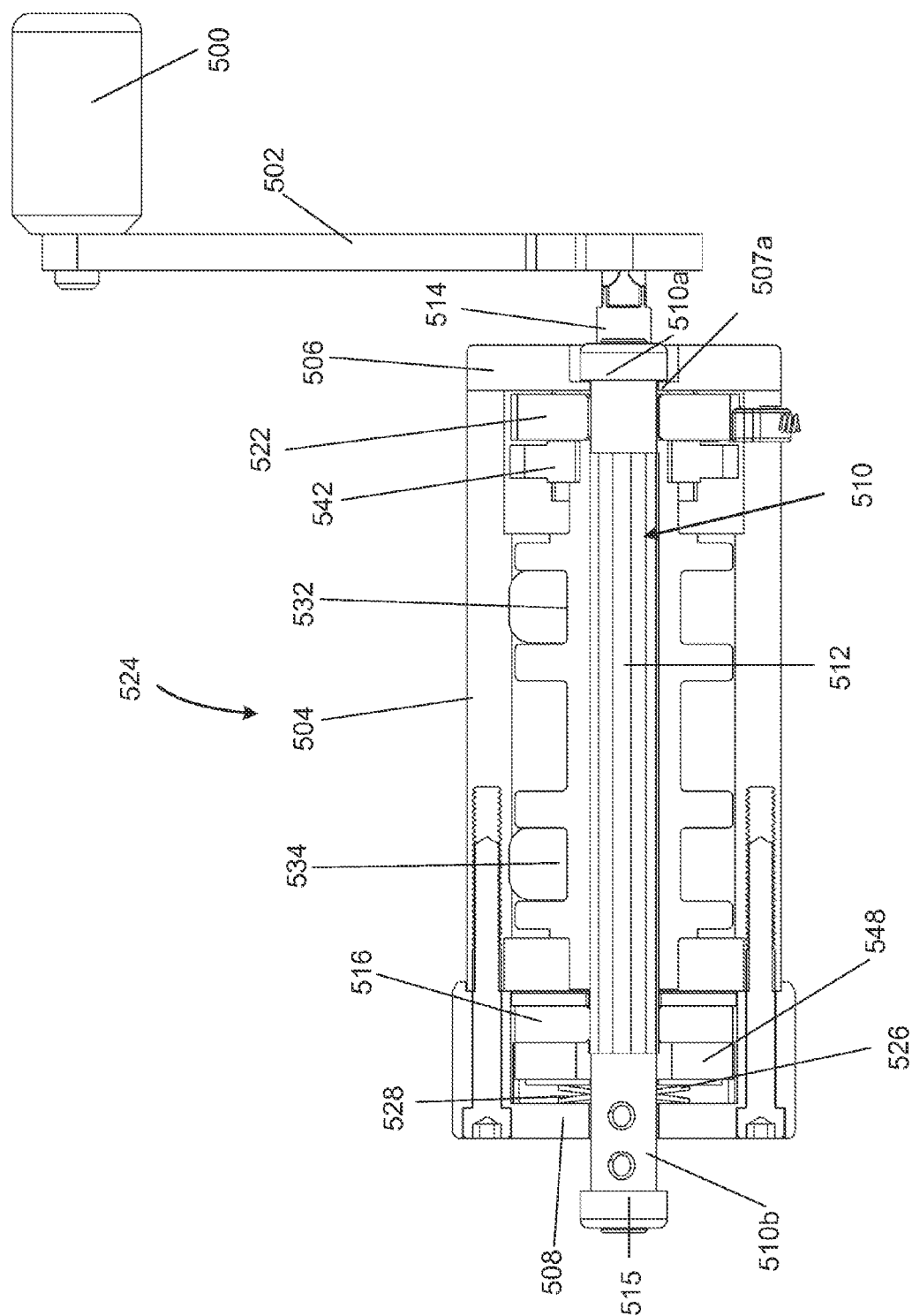


FIG. 22

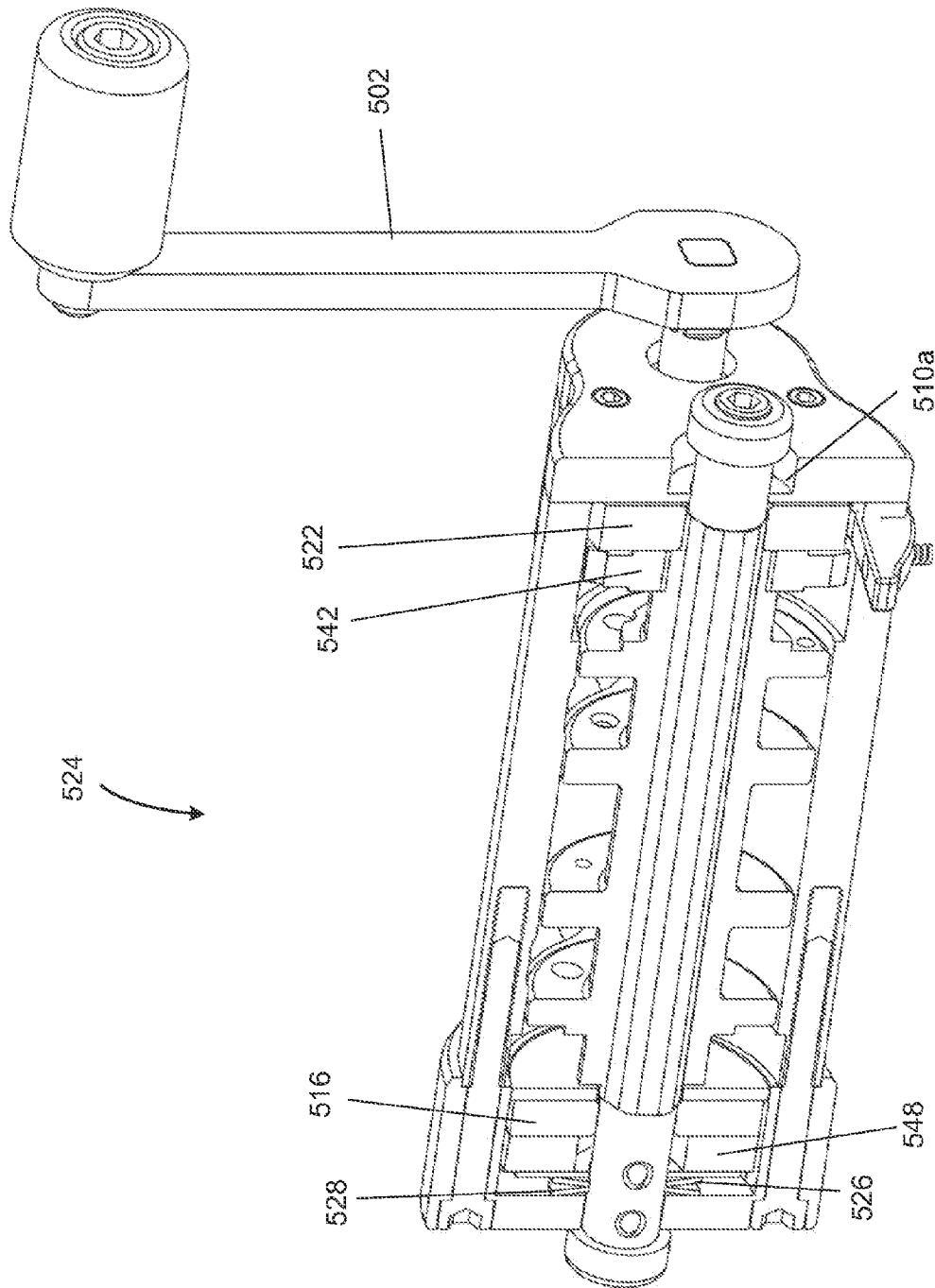


FIG. 23

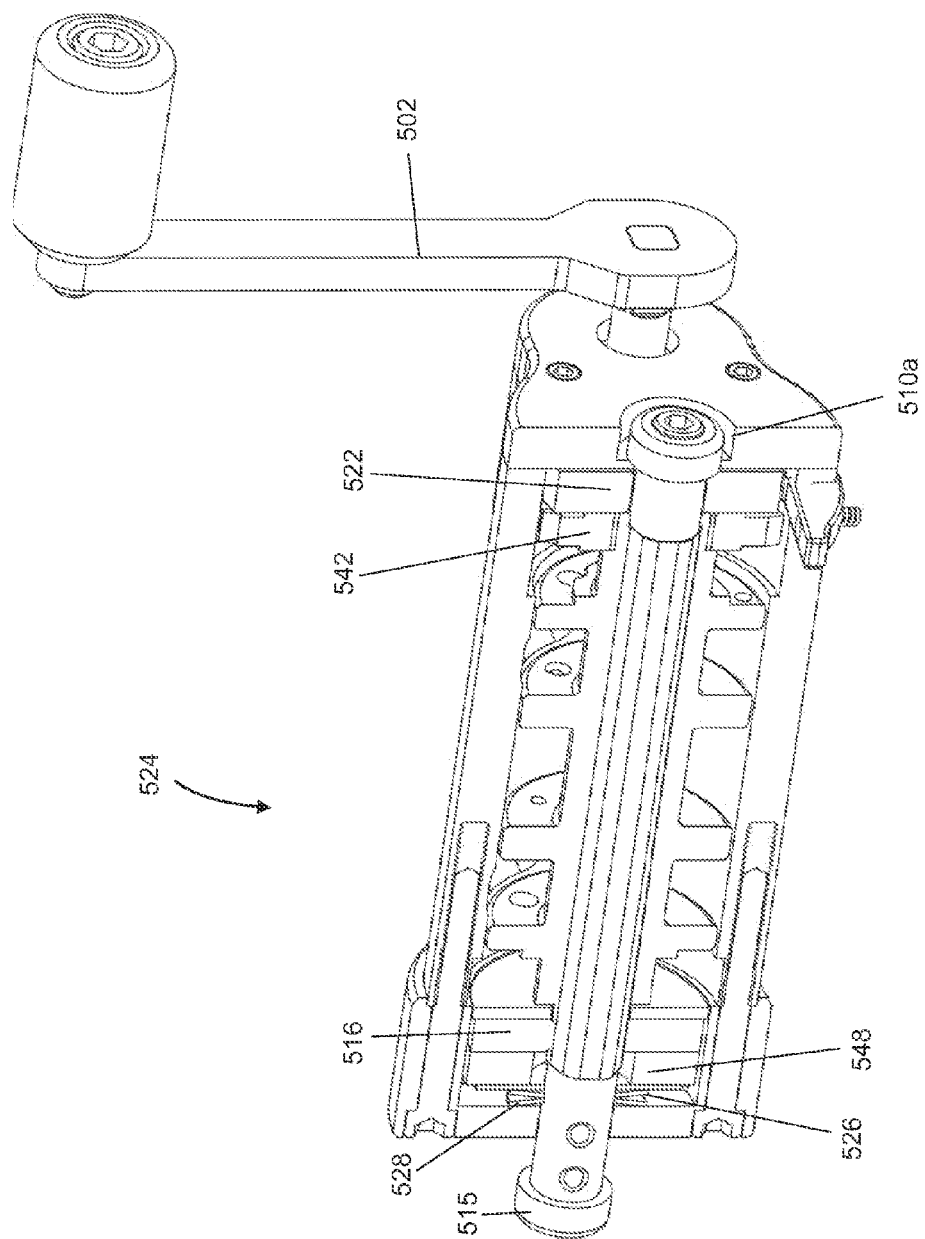


FIG. 24

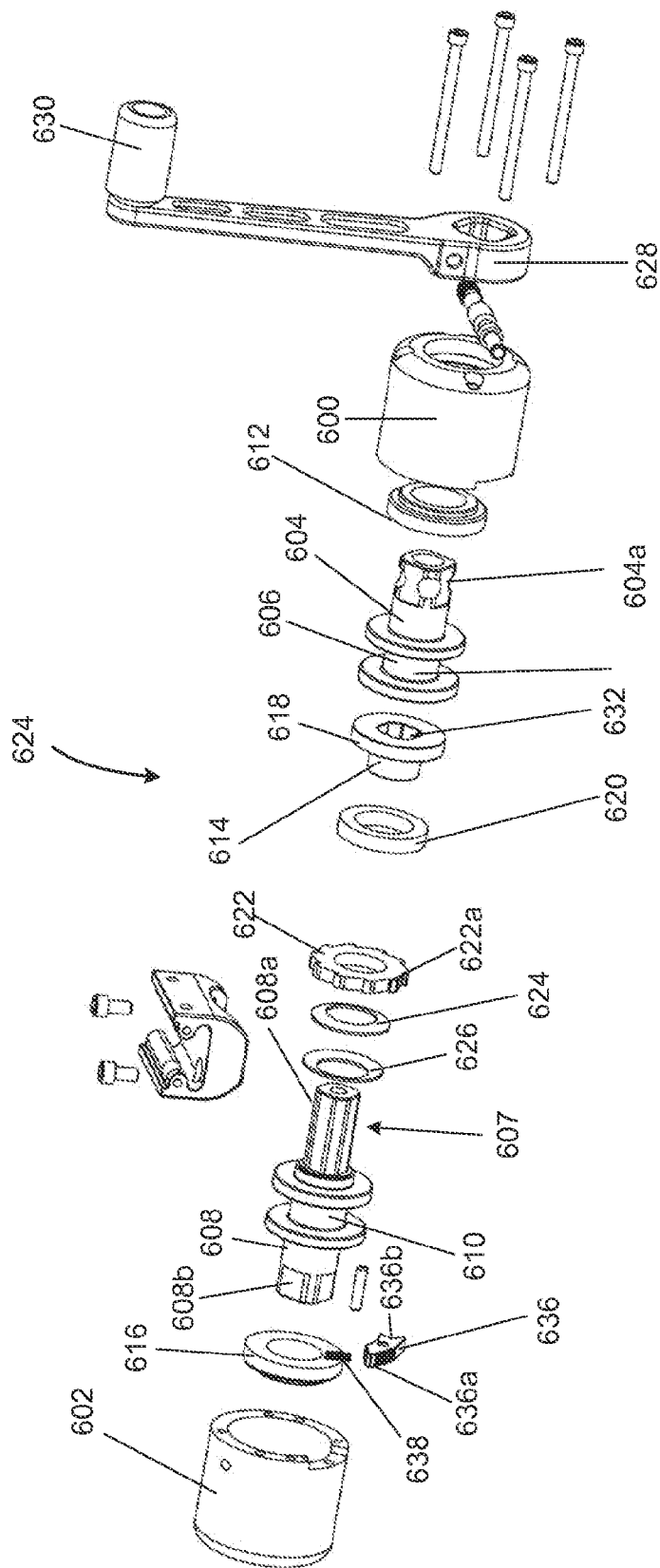
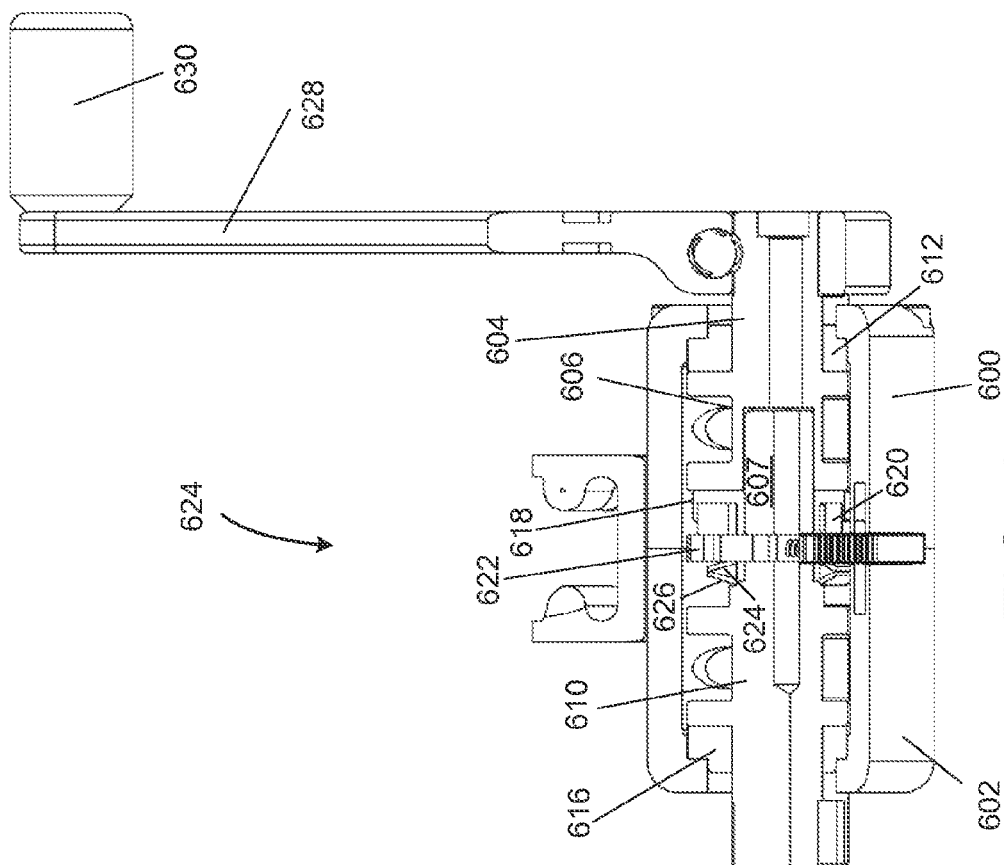
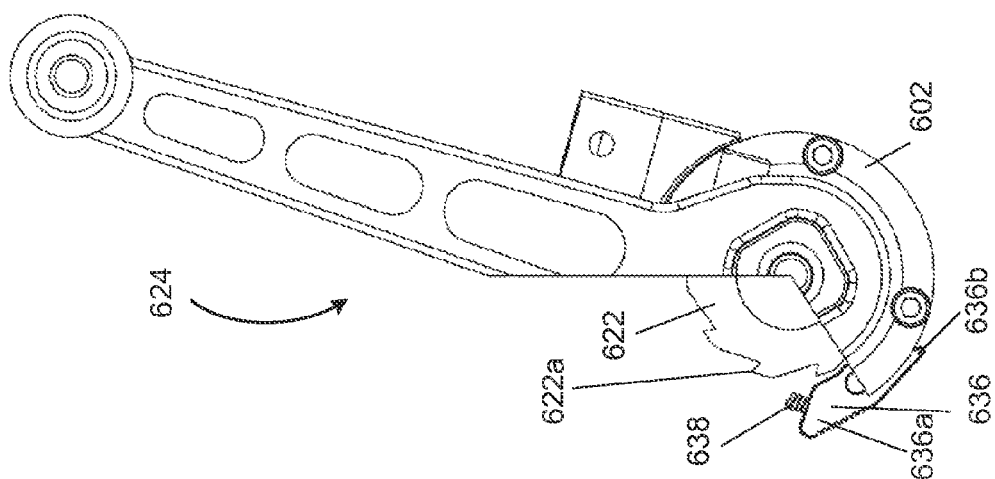


FIG. 25



FLG. 27



26^x
G
F

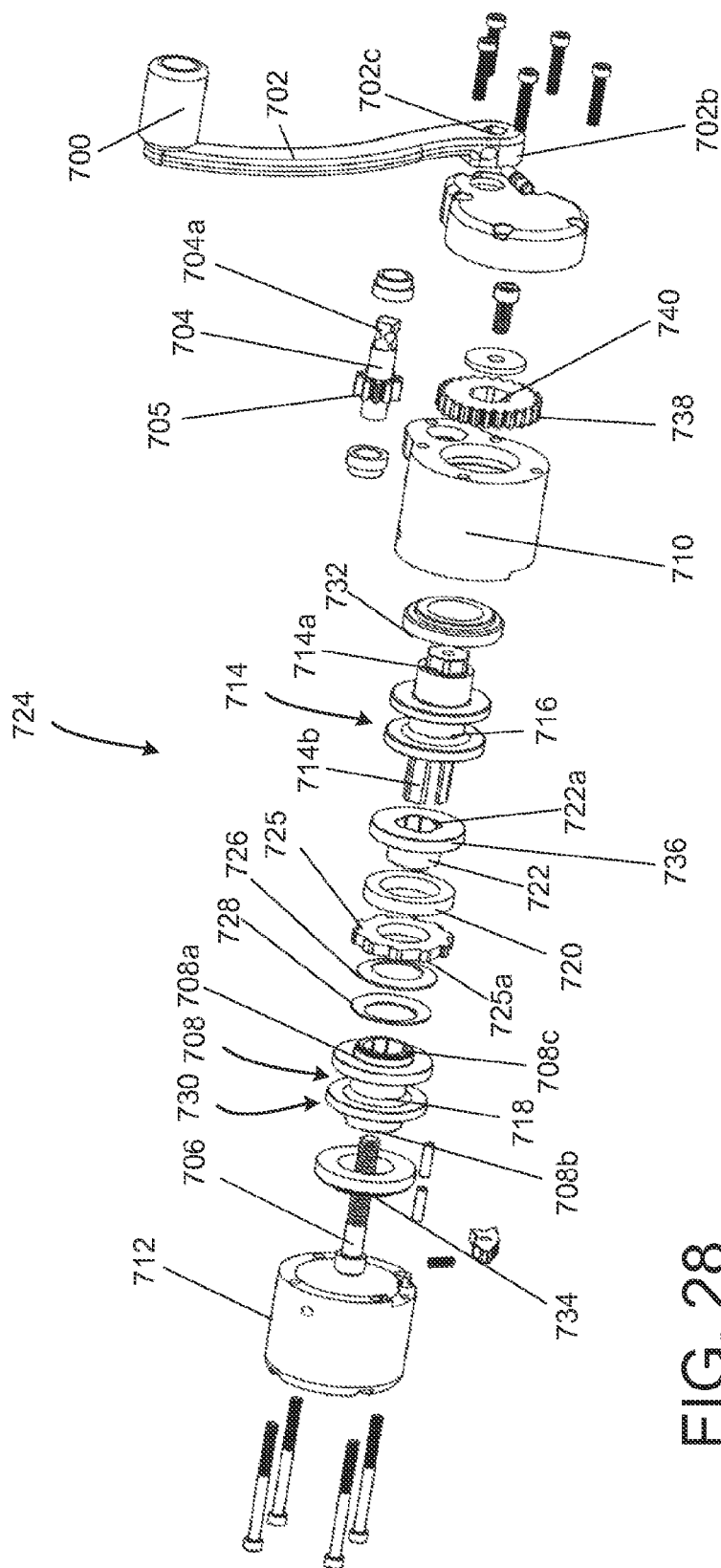


FIG. 28

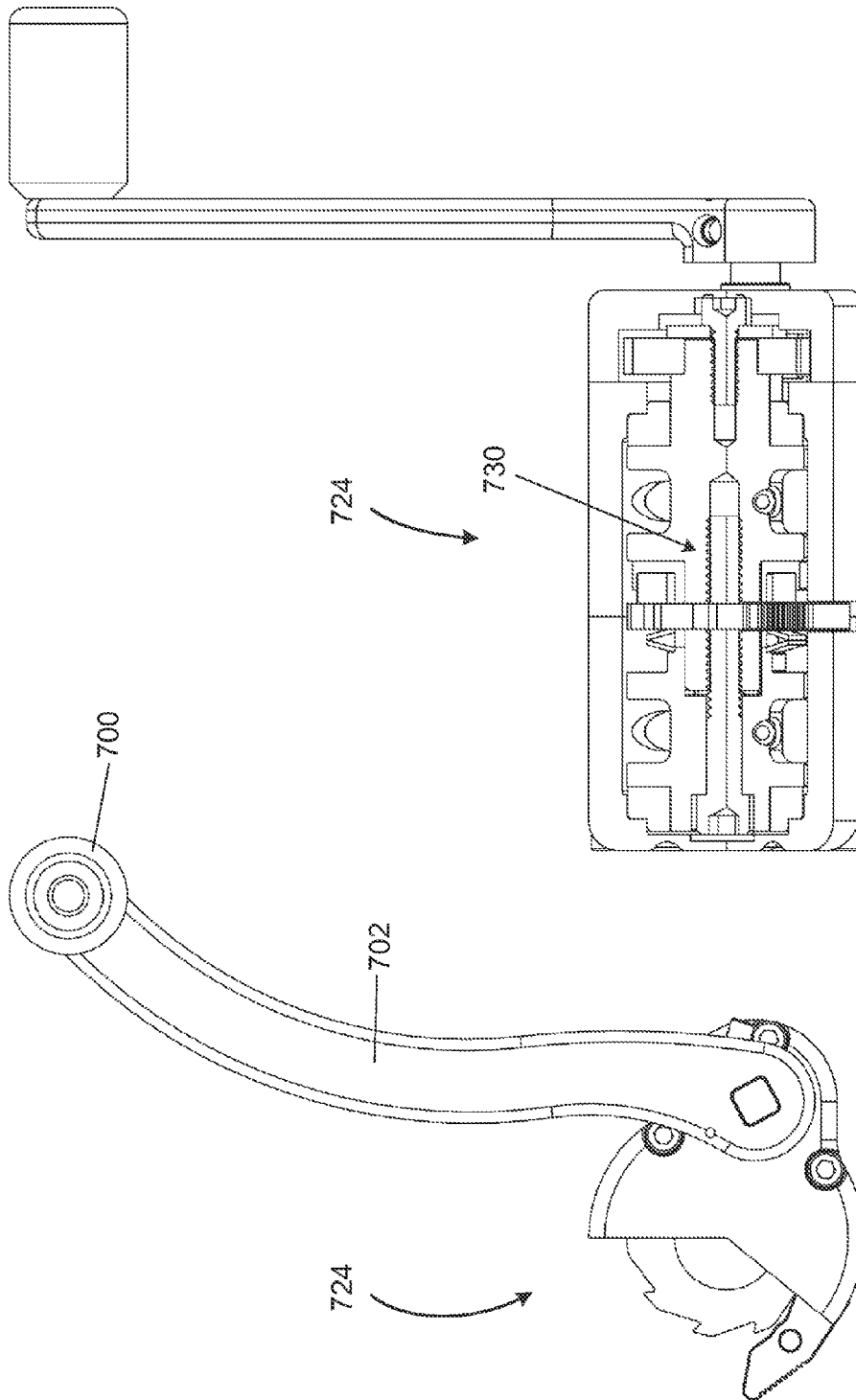


FIG. 30

FIG. 29

1

CROSSBOW WITH A CRANK COCKING AND RELEASE MECHANISM

BACKGROUND

The present invention relates generally to crossbows and in particular to a release mechanism for cocking and un-cocking the bowstring on a crossbow.

Crossbows have been used since the middle ages. Crossbows have evolved to include cams and synthetic split limbs that greatly increase firing velocity. Because of the increased forces exerted by the limbs, cranks have been used to cock the bowstring. One problem encountered when using a crank mechanism to cock the bowstring arises when the user wishes to disconnect the cranking mechanism from the bowstring. Moreover, increased firing velocity also creates a problem when a crossbow is dry-fired in order to release the bowstring from a cocked position into an un-cocked position without firing a bolt or arrow. Unloaded or dry firing impacts can damage the bowstring, limbs, cams and other components. Dry firing also creates a safety concern. The designs disclosed herein seek to address many of the concerns that arise with today's crossbows.

SUMMARY OF THE INVENTION

In one embodiment, a bowstring drawing mechanism for use on a weapon comprises (1) a generally cylindrical housing; (2) a shaft; (3) a ratchet wheel that has a toothed outer circumferential surface and a splined inner circumferential surface; (4) at least one rope spool received on, axially moveable with respect to, and rotationally fixed to, the shaft; (5) a clutch mechanism received on the shaft intermediate the shaft threaded second end and the shaft axial splines; and (6) a handle operatively coupled to the shaft. The shaft has (1) a first end; (2) a threaded second end; (3) an axis extending between the first and second ends; and (4) axial splines formed on an outer circumference of the shaft intermediate the first and second ends. The axial splines of the shaft are each substantially parallel to the shaft axis. A portion of the shaft is rotatably mounted in the housing. The ratchet wheel is positioned on the shaft proximate to the first end of the shaft. The rope spool is configured to attach to a first and a second end of a rope and the rope is configured to be releasably attached to a bowstring. The clutch mechanism has splines formed on an inner circumferential surface. When the shaft is in a first axial position, with respect to the clutch mechanism and the ratchet wheel, the shaft splines are engaged with the ratchet wheel splines and are disengaged from the clutch mechanism splines so that the ratchet wheel allows the shaft to rotate in a first direction and prevents the shaft from rotating in a second opposite direction. Additionally, when the shaft is in a second axial position, with respect to the clutch mechanism and the ratchet wheel, the shaft splines are disengaged from the ratchet wheel splines and engaged with the clutch mechanism splines so that the clutch prevents the shaft from rotating in the first direction and the second direction until force is applied to the handle.

In still another embodiment, a bowstring drawing mechanism comprises (1) a generally cylindrical housing having a first end and a second end; (2) a shaft rotatably mounted in the housing has a first end that extends through the housing first end and second end; (3) at least one rope spool received on, and rotationally fixed to, the shaft intermediate the shaft first and second ends; (4) a handle operatively coupled to the shaft; and (5) a clutch mechanism received on the shaft. At least one of the rope spools is configured to attach to a first and a second

2

end of a rope that is configured to be releasably attached to a bowstring. The clutch mechanism is configured to at least temporarily rotationally fix the shaft to the housing to prevent the shaft from rotating in a first direction and an opposite second direction until a force is exerted on the handle.

In yet another embodiment, a bowstring drawing mechanism comprises (1) a generally cylindrical housing having a first end and a second end; (2) a shaft rotatably mounted in the housing and having a first end that extends through the housing first end, and a second end; (3) a first rope spool received on, and rotationally fixed to, the shaft intermediate the shaft first and second ends; (4) a second rope spool received on, and rotationally fixed to, the shaft intermediate the first rope spool and the shaft second end; (5) a handle operatively coupled to the shaft; and (6) a clutch mechanism received on the shaft intermediate the first and second rope spools. In various embodiments, the first and second rope spools are configured to respectively attach to a first and a second end of a rope that is configured to be releasably attached to a bowstring and the clutch mechanism is configured to at least temporarily rotationally couple the shaft to the housing.

In some embodiments, the clutch mechanism further comprises a disk rotationally fixed to the shaft, a ratchet wheel rotatably received on the shaft, and a clutch plate positioned intermediate the disk and the ratchet wheel. In other embodiments, the bowstring drawing mechanism further comprises a first gear received on and rotationally fixed to the shaft and a second shaft rotatably mounted in the housing where the second shaft has a first end coupled to the handle, and a second gear rotationally fixed to the second shaft. In some of these embodiments, the first gear is operatively coupled to the second gear so that rotation of the handle in a first direction causes the second shaft and the second gear to rotate in the first direction and the first gear and the first shaft to rotate in the opposite direction.

BRIEF DESCRIPTION OF THE DRAWINGS

Various embodiments of an apparatus, system, and method for monitoring sports performance are described below. In the course of this description, reference will be made to the accompanying drawings, which are not necessarily drawn to scale, and wherein:

FIG. 1 is a perspective view of an embodiment of a crossbow;

FIG. 2 is a right side plan view of the crossbow of FIG. 1;

FIG. 3 is an exploded view of a bowstring cocking device for use with the crossbow of FIG. 1;

FIG. 4 is a perspective view of the bowstring cocking device of FIG. 3;

FIG. 5 is a partial sectional view of the bowstring cocking device of FIG. 3;

FIG. 6 is a partial sectional view of the bowstring cocking device of FIG. 3;

FIG. 7 is a partial perspective view of the bowstring cocking device of FIG. 3, in a first position;

FIG. 8 is a partial perspective view of the bowstring cocking device of FIG. 3, in a second position;

FIG. 9 is an exploded view of an embodiment of a bowstring cocking device for use in the crossbow of FIG. 1;

FIG. 10 is a partial sectional view of the bowstring cocking device of FIG. 9;

FIG. 11 is an exploded view of an embodiment of a bowstring cocking device for use in the crossbow of FIG. 1;

FIG. 12 is a partial sectional view of the bowstring cocking device of FIG. 11, in a first position;

3

FIG. 13 is a partial sectional view of the bowstring cocking device of FIG. 11, in a second position;

FIG. 14 is an exploded view of an embodiment of a bowstring cocking device for use in the crossbow of FIG. 1;

FIG. 15 is an exploded view of an embodiment of a bowstring cocking device for use in the crossbow of FIG. 1;

FIG. 16 is partial sectional view of the bowstring cocking device of FIG. 15;

FIG. 17 is partial perspective view of the bowstring cocking device of FIG. 15;

FIG. 18 is an exploded view of an embodiment of a bowstring cocking device for use in the crossbow of FIG. 1;

FIG. 19 is partial sectional view of the bowstring cocking device of FIG. 18;

FIG. 20 is an exploded view of an embodiment of a bowstring cocking device for use in the crossbow of FIG. 1;

FIG. 21 is a partial sectional view of the bowstring cocking device of FIG. 20, in a first position;

FIG. 22 is a partial sectional view of the bowstring cocking device of FIG. 20, in a second position;

FIG. 23 is a partial perspective view of the bowstring cocking device of FIG. 20, in a first position;

FIG. 24 is a partial perspective view of the bowstring cocking device of FIG. 20, in a second position;

FIG. 25 is an exploded view of an embodiment of a bowstring cocking device for use in the crossbow of FIG. 1;

FIG. 26 is a partial sectional view of the bowstring cocking device of FIG. 25;

FIG. 27 is partial sectional view of the bowstring cocking device of FIG. 25;

FIG. 28 is an exploded view of an embodiment of a bowstring cocking device for use in the crossbow of FIG. 1;

FIG. 29 is a partial sectional view of the bowstring cocking device of FIGS. 28; and

FIG. 30 is partial sectional view of the bowstring cocking device of FIG. 28.

DETAILED DESCRIPTION

Various embodiments now will be described more fully hereinafter with reference to the accompanying drawings. It should be understood that the invention may be embodied in many different forms and should not be construed as limited to the embodiments set forth herein. Like numbers refer to like elements throughout.

Overview

Referring to FIGS. 1 and 2, a crossbow 10 is shown having a barrel 12 that has a first end 14 coupled to a riser 16 and a second end 18 coupled to a pistol grip 20. A stock 22 is coupled to the elongated barrel second end and terminates at a butt 26. In the embodiment shown, the stock length is adjustable, but in other embodiments the stock may have a fixed length. A grip 28 is coupled to the barrel 12 intermediate the first and second ends 14 and 18. A retention spring 30 is operatively coupled to a top surface 32 of the barrel 12. A first limb 36 has a first end 36a operatively coupled to a left side 38 of the riser 16 and a second end 36b operatively coupled to a bowstring 44. A second limb 40 has a first end 40a that is operatively coupled to a right side 42 of the riser 16 and a second end 40b that is operatively coupled to the bowstring 44. A trigger mechanism 46 is used to fire the crossbow 10 when the bowstring 44 is in a cocked position.

A bowstring cocking device 24 is releasably coupled to the bowstring 44 and contains a crank mechanism 48, an elongated cocking rope 50 having a first hook 50a and a second hook 50b. The first and second hooks 50a and 50b are configured to attach to the bowstring 44 so that when a user turns

4

the crank 18, the cocking rope 50 pulls the bowstring 44 into a cocked position. In general, the bowstring cocking device 24 is used to move the bowstring 44 into a cocked firing position and/or to move the bowstring 44 from the cocked firing position back into a resting position without having to fire or dry fire the crossbow 10. Various embodiments use a ratchet wheel and detent to allow the crank to turn in a winding first direction while preventing the crank from turning in an unwinding second direction. In various embodiments, when a clutch in the crank is engaged and the ratchet and detent disengaged, the clutch allows the crank to rotate in the unwinding second direction in a controlled manner if the user wishes to either provide slack in the elongated cocking rope 50 to remove the first and second hooks 50a and 50b from the bowstring 44 or to move the crossbow bowstring 44 into the resting position without firing or dry firing the crossbow 10.

First Embodiment of Bowstring Cocking Device

FIGS. 3-8 illustrate a first embodiment of a bowstring cocking device 24 having a dual shaft mechanism, a ratchet wheel, and a clutch mechanism.

Bowstring Cocking Device Structure

Referring to FIGS. 3 and 4, one embodiment of a bowstring cocking device 24 is shown. In particular, the bowstring cocking device 24 comprises a handle 100, a crank 102, a housing body 104 having a first and second cover 106 and 108, a shaft 110, a spool body 120, a ratchet wheel 122, a gear 142, a clutch mechanism 148, and a knob 152.

Housing Body

The first cover 106 and the second cover 108 together enclose the various parts of the bowstring cocking device 24 within the housing body 104. The housing body 104 is generally cylindrical in shape, but may be formed in any suitable shape. In various embodiments, the housing 104 may be formed from any suitable material (e.g., alloy, stainless steel, ceramic, polymers, etc.) and in particular embodiments the housing 104 is formed from aluminum. A first mounting bracket 162 and a second mounting bracket 164 are coupled to the housing body 104 at a top surface and are configured to mount the bowstring cocking device 24 to the underside of the crossbow 10, as shown in FIGS. 1 and 2. In various embodiments, the mounting brackets may be connected to the crossbow 10 using any suitable fastener (e.g., a bolt, a pin, a rivet, weldments, etc.).

Shaft

Referring to FIG. 3, the shaft 110 has a first end 110a, a second end 110b and a splined portion 112 intermediate the first and second ends 110a and 110b. In addition to the splined portion 112, a threaded shaft portion 114 is positioned proximate the shaft second 110b. The shaft 110 also has a smooth portion 115 positioned intermediate the splined portion 112 and the threaded portion 114. The splined shaft 110 is generally cylindrically shaped with the axial splines 112a formed on an outer circumference of the shaft intermediate to the first end 110a and the second end 110b where each axial spline is substantially parallel to the axis of the shaft 110.

The shaft 110 is rotatably mounted in the housing 104 so that the shaft first end 110a extends through a hole 107 formed through the first cover 106 and the shaft second end 110b extends through the second cover 108. The splines 112a are configured to interact with a splined ring 116, a splined inner circumference bore 136 of the rope spool body 120, and a splined inner circumferential surface 122a of the ratchet wheel 122. The second end 110b of the shaft 110 passes through the second cover 108 of the housing body 104 where the threaded end 114 is configured to receive a first spring 154, a threaded knob 152, a second spring 156, and a knob bracket 130.

Rope Spool Body

The rope spool body **120** comprises a first end **120a** and a second end **120b**. The rope spool body **120** is generally cylindrical in shape and has a first rope spool **132** and a second rope spool **134**. The rope spools **132**, **134** provide a space for the cocking rope **50** to wind around when the bowstring **44** is pulled from the resting position into the cocked position. The rope spool body **120** is received on, axially moveable with respect to, and rotationally fixed to the shaft **110** by the splined bore **136** formed through the rope spool body **120**. In addition, the rope spools **132** and **134** are configured to operatively attach to a first and a second end of the cocking rope **50**. A first bearing **138** encompasses the first end **120a** of the rope spool body **120** and a second bearing **140** encompasses the second end **120b**. The bearings **138**, **140** help the rope spool body **120** rotate smoothly when the rope spool body **120** is rotated within the housing body **104**. Referring to FIG. 6, a first opening **124** and a second opening **126** in the housing body **104** are positioned above the first rope spool **132** and the second rope spool **134**, respectively, and allow the cocking rope **50** (not shown) to enter the housing body **104** and wind around the rope spools **132**, **134**.

Gear

Referring again to FIG. 3, the gear **142** is rotationally fixed to the first end **120a** of the rope spool body **120** by screws (not numbered). In various embodiments, the gear **142** may be integrally formed with the rope spool body **120**. In still other embodiments, the gear **142** may be rotationally fixed to the shaft **110** via a splined engagement similar to the splined engagement between the rope spool body **120** and the shaft splines **112a**.

Ratchet Wheel and Detent

The first cover **106** of the housing body **104** has a circular flange **109** that defines a recess (not shown) that is configured to receive the ratchet wheel **122** therein. The ratchet wheel **122** has (1) the splined inner circumference surface **122a** and is received on the shaft splined portion **112**, and (2) a toothed outer circumferential surface **122b**. The ratchet wheel **122** is positioned on the shaft **110** proximate to the shaft first end **110a**. A spring **144** is operatively received in the housing **104** such that one end of the spring **144** engages a first end **146** of a button **160** that is moved into and out of engagement with the ratchet wheel toothed outer circumferential surface **122b** to prevent unwanted rotation of the ratchet wheel **122**. The button **160** is operatively engaged with the spring **144** and allows a user to move one end of the button **146** into and out of engagement with the ratchet wheel toothed outer circumferential surface **122b** against the bias of the spring **144**. Referring to FIG. 5, the pin **160** is biased radially inward by the spring **144**.

Clutch Mechanism

Referring again to FIGS. 3 and 6, the clutch **148** is received on the shaft **110** adjacent the shaft smooth portion **115**, which is intermediate the threaded shaft second end **110b** and the shaft splined portion **112**. The clutch **148** is also received in, and rotationally fixed to, the second cover **108**. That is, the second cover **108** contains a substantially square area that receives the substantially square clutch mechanism **148** so that the clutch mechanism is rotationally fixed to the second cover **108**. It should be understood that the clutch mechanism **148** may be rotationally fixed to the second cover **108** by other suitable means. The clutch **148** has a smooth, circular inner circumferential surface **150** for receiving the shaft smooth portion **115**. In various embodiments, the clutch **148** is generally square shaped and made from Friction-modifying

materials (e.g., Kevlar, metal, alloy, semi-metallic material, sintered metal, resin, carbon material, or woven glass material).

Adjustment Knob

The adjustment knob **152** is generally circular in shape and has a threaded inner circumferential surface that is configured to engage with the threaded portion **114** of the shaft second end **110b**. The adjustment knob **152** is generally positioned on the shaft threaded portion **114** intermediate the second cover **108** and the knob bracket **130**. A first spring **154** is positioned intermediate the adjustment knob **152** and the second cover **108** and a second spring is positioned intermediate the adjustment knob **152** and the adjustment knob bracket **130**. The first and second springs **154** and **156** assist in biasing the shaft **110** in the axial direction depending on the position of the adjustment knob **152**, which in turn causes the splined shaft **110** to either be rotationally fixed or rotatable with one of the ratchet wheel **122** or splined ring **116** depending on the shaft's position within the following parts: the splined ring **116**, the spool body **120**, the ratchet wheel **122**, and the threaded knob **152**.

Crank Shaft, Gear and Handle

Referring once again to FIG. 3, the handle **100** is coupled to a first end **102a** of the crank **102** by a bolt **101**. In various embodiments, the crank **102** may be integrally formed with the handle **100**, or in other embodiments, the crank **102** may be connected to the handle **100** using any suitable fastener (e.g., a bolt, a pin, a rivet, weldments, etc.). The crank **102** and the handle **100** are generally perpendicular to each other when attached. The crank **102** also has a second end **102b** with an opening **102c** that is configured to operatively engage with a gear crank shaft **111**. The gear crank shaft **111** is rotationally fixed to the crank **102** since the shape of the crank opening **102c** matches the shape of an end **111a** of the gear crank shaft **111**. That is, the crank opening **102c** and the end **111a** of the gear crank shaft **111** are both substantially square in shape. In various embodiments, the crank **102** may be coupled to the gear crank shaft **111** in any suitable manner (e.g., a bolt, a pin, a rivet, a cotter pin, weldments, etc.).

The gear crank shaft **111** is operatively coupled to a crank gear **113** so that the gear crank shaft is rotationally fixed to the crank gear. In various embodiments, the crank gear **113** is integrally formed with the gear crank shaft **111**. In various other embodiments, the crank gear **113** may be connected to the gear crank shaft **111** using any suitable fastener (e.g., a bolt, a pin, a rivet, a cotter pin, weldments, etc.). The crank gear **113** has teeth that match the teeth of the gear **142**.

Parts Enclosed Inside the Housing Body

Referring to FIG. 4, when the bowstring cocking device is assembled, the following parts fit inside the housing body **104** beginning at the first cover **106** and progressing along to the second cover **108**: the ratchet wheel **122**, the gear **142**, the rope spool body **120**, the clutch mechanism **148**, and running through the length of the inner circumference of each of these parts is the shaft **110**. Thus, in the present embodiment, the bowstring cocking device **24** has at least a portion of the following parts exposed: the handle **100**, the crank **102**, the crank gear shaft **111** exposed through the opening **102c** of the crank **102**, the first cover **106**, the first end of the splined shaft **110a**, the housing body **104**, the second cover **108**, and the knob bracket **130** enclosing the knob **152**, the first spring **154** (not shown), and the second spring **156**.

Bowstring Cocking Device Operation

First Position

Referring to FIG. 7, the bowstring cocking device **24** is shown in a first position where the shaft **110** is moved axially into a first position with respect to the housing body **104** such

7

that the splined portion 112 is engaged with the ratchet wheel splined inner circumferential surface 122a and is disengaged from the splined ring 116. In the first position, the splined ring 116 is positioned on the shaft smooth portion 115 so that the splined portion 112 is disengaged from the splined ring 116. The bowstring cocking device 24 may be moved into the first position as shown in FIG. 7 by turning the adjustment knob 152. Because the knob 152 has an inner circumferential surface that is threaded, the threaded engagement of the knob 152 with the threaded end 114 of the splined shaft 110 causes the shaft 110 to move axially (toward the right with regard to FIG. 7) so that the splined portion 112 engages with the inner circumferential splined surface of the ratchet wheel 122a, which causes the ratchet wheel 122 to be rotationally fixed with the shaft 110 while the splined ring 116 is allowed to rotate with respect to the shaft 110.

Referring again to FIG. 5, the teeth of the crank gear 113 engage with the teeth of the gear 142 so that rotation of the crank gear 113 in the clockwise direction causes the gear 142 to rotate in a counterclockwise direction (with respect to the view of FIG. 5). Thus when the crank 102 (FIG. 6) is turned clockwise, the gear crank shaft 111 also rotates clockwise in turn causing the crank gear 113 to rotate clockwise. This, in turn, rotates the gear 142 in the opposite, counter-clockwise, direction. Because the gear 142 is rotationally fixed to the shaft 110 via the spool body 120, and the ratchet wheel 122 and the spool body 120 are rotationally fixed to the shaft 110, rotation of crank 102 and crank shaft 111 clockwise causes the spool body 120 to rotate counterclockwise. As a result, as the rope spool body 120 rotates counterclockwise, it winds up the cocking rope 50 and pulls the bowstring 44 into the cocked position. Because the spring 144 biases the pin 146 into the ratchet wheel toothed outer circumferential surface 122b, the pin 146 prevents unwanted rotation of the ratchet wheel 122 in the clockwise direction. As a result, the spool body is also prevented from rotating in the clockwise direction since the ratchet wheel 122 and the spool body 120 is rotationally fixed to the shaft 110. In this way, the spool body 120 will not inadvertently rotate clockwise in response to the bias exerted on the bowstring cocking rope 50 by the bowstring 44.

Second Position

FIG. 8 shows the shaft 110 in a second position where the shaft splined portion 112 is disengaged from the ratchet wheel 122 and is engaged with the splined ring 116. In the second position, either the bowstring cocking rope 50 may be slightly released so that the first and second hooks may be removed from the bowstring once the bowstring is in the cocked position, or (2) the bowstring 44 may be moved from the firing position into the resting position without firing or dry firing the crossbow 10. The bowstring cocking device is moved into the second position when the user rotates the knob 152 so that the threaded engagement between the knob 152 and the threaded shaft portion 114 causes the shaft to move axially rearward (e.g., to the left in FIG. 8) so that the shaft splined portion 112 engages the splined ring 116 and disengages from the splined ratchet wheel 122.

Thus, when the bowstring cocking device 24 is in the second position, the ratchet wheel 122 no longer prevents the shaft 110 from rotating in the clockwise direction since the ratchet wheel 122 is no longer rotationally fixed to the shaft 110. Instead, the engagement of a face of the splined ring 116 with a face of the clutch mechanism 148 prevents the shaft 110 from spinning. That is, the frictional force between the face of the splined ring 116 and the corresponding face of the clutch mechanism 148 is greater than the pulling force exerted on the bowstring cocking rope 50 by the bowstring 44. As a result, the user may turn the handle 100 and the crank 102

8

in the counterclockwise direction in a controlled manner, which in turn causes the shaft 110 to rotate in the clockwise direction, thereby letting the rope 50 out from the rope spools 132, 134. In this way, the user can either release tension on the bowstring cocking rope to allow the user to remove the first and second hooks 50a and 50b from the bowstring 44 or to release the bowstring 44 from the cocked position into the resting position in a controlled manner without firing or dry firing the crossbow.

Bowstring Cocking Device Alternate Embodiment

Referring to FIGS. 9 and 10, an alternative embodiment of the bowstring cocking device 24 is shown having a single shaft design as opposed to the dual shaft design shown in the embodiment of FIGured 3-8. Thus, for purposes of ease of understanding and clarity, only certain parts will be discussed to highlight the differences in the structure and operation of the embodiment shown in FIGS. 9 and 10 as compared to the embodiment shown in FIGS. 3-8. The handle 100 and crank shaft 102 couples directly to the shaft 110. That is, the crank second end 102b contains a square opening 102c that mates with, and is rotationally fixed to, the first end 110a of the shaft 110. Additionally, this alternate embodiment utilizes a concave flat spring 166 that is positioned intermediate the second cover 108 and the clutch mechanism 148 to bias the clutch mechanism 148 into the splined ring 116. Similar to the embodiment shown in FIGS. 3-8, the shaft 110 is moveable between a first position in which the shaft splined portion 112 is engaged with the splines funned on the inner circumferential surface of the ratchet wheel 122 and disengaged from the splined ring 116, and the second position in which the shaft splined portion is engaged with the splined ring 116 and disengaged from the ratchet wheel 122 by rotating the adjustment knob 152 as described above.

Second Embodiment of Bowstring Cocking Device

FIGS. 11-13 illustrate a second embodiment of a bowstring cocking device 224 having a housing body 200, a single shaft 214, first and second rope spools 216 and 218, a clutch mechanism 230, a sleeve 236 and a crank 248.

Bowstring Cocking Device Structure

Housing Body

As shown in FIG. 11, the housing body 200 has a first end 200a and a second end 200b. Coupled to the first end of the housing body 200a is a first cover 202. Coupled to the second end of the housing body 200b is a second cover 204. The first cover 202 and the second cover 204 are rotationally fixed respectively to the housing body first end 200a and the housing body second end 200b by screws (not numbered) or other suitable fasteners such as bolts, rivets, weldments, etc. The first cover 202 and the second cover 204 enclose the various parts of the bowstring cocking device 224 within the housing body 200. The housing body 200 is generally cylindrical in shape, but may be formed in any suitable shape. In various embodiments, the housing body 200 may be formed from any suitable material (e.g., alloy, stainless steel, ceramic, polymers, etc.) and in particular embodiments the housing body is formed from aluminum. The housing body 200 also has a first mounting bracket 206 and a second mounting bracket 208 for attaching the bowstring cocking device 224 to the crossbow 10. Referring to FIGS. 12 and 13, the housing body 200 has a first opening 210 and a second opening 212 for allowing the bowstring cocking rope 50 to pass into the housing body 200.

Shaft, Rope Spools, and Bearings

As shown in FIG. 11, the bowstring cocking device 224 has a shaft 214 that has a first end 214a and a second end 214b. The shaft 214 is generally cylindrical in shape. The shaft 214 also has a first rope spool 216 and a second rope spool 218 received thereon proximate to the shaft second end 214b. The

first spool **216** has a first end **216a** and a second end **216b**. In addition, the second spool **218** has a first end **218a** and a second end **218b**. Proximate to the first end of the shaft **214a** is an elongated opening **220** for receiving a pin **222** as explained in more detail herein. The pin **222** is generally cylindrical in shape and may be formed from any suitable material such as aluminum, stainless steel, etc. The rope spools **216**, **218** are received on, and rotationally fixed to, the shaft **214** intermediate the shaft first end **214a** and the shaft second end **214b**. The rope spools **216**, **218** are configured to attach to a first and a second end of the bowstring cocking rope **50**, which is configured to be releasably attached to the bowstring **44**. In various embodiments, the spools **216**, **218** are integrally formed with the shaft. In other embodiments, the first and second spools **216**, **218** are attached to the shaft **214** using any suitable fastener (e.g., welded, screws, rivets, threaded attachment, etc.).

A first bearing **226** is received around the outer circumferential surface of the first spool first end **216a**. Additionally, a second bearing **228** is received around the outer circumferential surface of the second spool second end **218b**. The first and second bearings **226** and **228** allow the shaft to be mounted coaxially with a central axis (not shown) of the housing body **200** while allowing the shaft to rotate freely within the housing body.

Clutch Mechanism

A clutch mechanism **230** is positioned on the shaft **214** proximate to the first rope spool first end **216a**. The clutch mechanism **230** is generally circular in shape and may be formed from friction-modifying materials (e.g., Kevlar, metal, alloy, semi-metallic material, sintered metal, resin, carbon material, or woven glass material). The clutch **230** has a circular inner circumference **232** for rotatably receiving the shaft **214** there through. The clutch **230** has a first tab **230a**, a second tab **230b**, a third tab **230c**, and a fourth tab **230d** that are used to rotationally fix the clutch mechanism **230** to the housing body **200**. That is, a recess **234** is formed at the first end of the housing body **200a** and is configured to receive the clutch mechanism **230**. The clutch tabs **230a**, **230b**, **230c**, **230d** fit into a respective first tab slot **234a** (not shown), second tab slot **234b**, third tab slot **234c**, and fourth tab slot **234d** (not shown) formed around the perimeter of the recess **234**. Thus, the clutch **230** is rotationally fixed to the housing body **200** by engagement of the tabs **230a**, **230b**, **230c**, **230d** and the tab slots **234a** (not shown), **234b**, **234c**, **234d** (not shown). In alternate embodiments, any number of tabs may be used to rotationally fix the clutch mechanism **230** to the housing body **200**. In other embodiments, the clutch mechanism **230** may be rotationally fixed to the housing body **200** by other suitable affixing means such as screws, rivets, pins, etc.

Sleeve, Sleeve Disk, and Springs

A sleeve **236** is received about the shaft first end **214a** intermediate to the clutch mechanism **230** and the first cover **202**. The sleeve **236** is generally oblong shaped and has a disk **238** coupled at the end proximate to the clutch **230**. The sleeve **236** has a substantially circular through hole **236a** for receiving the shaft **214** and is rotationally fixed to the shaft **214** by a pin **222** that is received through an opening **240** in the sleeve **236**. The disk **238** may be integrally formed with the sleeve **236** or it may be attached to the sleeve using any suitable connecting means such as screws, rivets, pins, press fit, weldments, etc. A first spring **242** and a second spring **244** are received about the sleeve **236** intermediate the disk **238** and the first cover **202** so that the first spring **242** abuts the first cover **202** and the second spring **244** abuts the disk **238**. The first spring **242** is concave towards the first cover **202**, while

the second spring **244** is concave towards the sleeve disk **238**. The springs **242** and **244** function to bias the sleeve **236** and disk **238** toward the clutch mechanism **230**.

Crank Shaft, Gear and Handle

Still referring to FIG. 11, the bowstring cocking device **224** comprises a handle **246** and a crank **248**. The handle **246** is coupled to a first end **248a** of the crank **248** by a bolt **250**. In various embodiments, the crank **248** may be integrally formed with the handle **246**, or in other embodiments, the crank **248** may be connected to the handle **246** using any suitable fastener (e.g., a bolt, a pin, a rivet, weldments, etc.). The handle **246** and the crank **248** are generally perpendicular to each other when attached to one another. The crank **248** also has a second end **248b** with an oblong first crank opening **252** that is configured to operatively engage with the oblong-shaped sleeve **236** to rotationally fix the sleeve **236** to the crank **248** since the shape of the first crank opening **252** matches the shape of the sleeve **236**. That is, the first crank opening **252** and sleeve **236** are both substantially oblong in shape where the first crank opening **252** is slightly larger than the sleeve **236**. In various embodiments, the crank **248** may be coupled to the sleeve **236** in any suitable manner (e.g., a bolt, a pin, a rivet, a cotter pin, weldments, etc.).

The crank also has a second opening **254** that is circular in shape, formed perpendicular to the first opening **252**, and is configured for receiving the pin **222**. When the pin **222** is inserted into the second opening **254**, it passes through the crank **248**, the circular opening **240** in the sleeve **236**, and the elongated opening **220** in the shaft **214** to rotationally fix the shaft **214** and the sleeve **236** to the crank **248**. Depending upon where the pin **222** is located in the elongated opening **220** of the shaft **214**, the shaft **214** may be rotationally fixed or rotatable within the housing body **200** as described in more detail below.

Bowstring Cocking Device Operation

First Position

Referring to FIG. 12, the bowstring cocking device **224** is shown in a first position where the crank **248** is substantially perpendicular to the shaft **214**. When the handle **246** is perpendicular to the housing body **200**, the sleeve disk **238** is pushed against the clutch material **230** by the springs **242** and **244** so that friction between a surface of the clutch mechanism and a surface of the disk **238** prevents the shaft **214** from rotating either clockwise or counterclockwise. Thus, when the bowstring cocking device **224** is in this first position, the frictional force between the clutch mechanism **230** and the sleeve disk **238** is greater than the pulling force exerted by the bowstring **44** on the spools **216**, **218** by the cocking rope **50**. As a result, the shaft **214** will not spin in either direction due to the frictional force.

Second Position

In FIG. 13, the bowstring cocking device **224** is shown in a second position where the handle **246** is pulled away from the housing body **200**, leaving the crank shaft **248** no longer perpendicular to the shaft **214**. In particular embodiments, the user can grab the handle **246** and pull it outward, away from the housing body first end **200a**, which causes the crank **248** to pivot about a pivot point **248c** and angle away from the housing body **200**. When the handle **246** and crank **248** are angled as shown in FIG. 13, the sleeve **236** and the disk **238** are moved axially toward the first cover **202** against the bias of springs **242** and **244** so that the disk **238** partially disengages from the clutch mechanism **230**. Because of the oblong opening **240** (FIG. 11) in the shaft **214**, the pin **222** can slide within the oblong opening **240** allowing the sleeve **236** and the sleeve disk **238** to slide slightly along the length of the shaft **214** relative to the clutch mechanism **230**. The sliding

11

action of the sleeve 236 and sleeve disk 238 releases the frictional force between the clutch mechanism 230 and the sleeve disk 238 allowing the sleeve 230, sleeve disk 238 and shaft 214 to rotate with respect to the housing body 200. Thus, the user can rotate the handle in either the counterclockwise or clockwise direction from the perspective of looking toward the handle from the right in FIG. 13. As a result, the user may turn the handle in the counterclockwise direction to move the bowstring 44 from the resting position into the cocked position. Moreover, once the bowstring is in the cocked position and is retained by the trigger mechanism, the user can release the tension on the bowstring 44 by the bowstring cocking rope 50 a sufficient amount to allow the user to remove the first and second hooks 50a and 50b from the bowstring 44. Furthermore, should the user want to move the bowstring 44 from the cocked position to an un-cocked position without firing or dry firing the crossbow, the user may do so by moving the bowstring cocking device 224 into the second position and turning the handle 246 and crank 248 clockwise.

Second Bowstring Cocking Device Alternate Embodiment

FIG. 14 illustrates an alternative embodiment of the bowstring cocking device 224 of FIGS. 11-13. For purposes of ease of understanding and clarity, only certain parts will be discussed to highlight the differences in the structure and operation of the embodiment shown in FIG. 14 as compared to the embodiment shown in FIGS. 11-13. In this embodiment, the clutch mechanism 230 shown in FIGS. 11-13 is replaced with a toothed ratchet wheel 256 that is received in the housing body 200 proximate to the housing body first end 200a. The ratchet wheel 256 is also received on the shaft first end 214a intermediate the first housing spool first end 216a and the shaft first end 214a. Similar to the clutch mechanism 230 in FIG. 11, the toothed ratchet wheel 256 in FIG. 14 has a first tab 256a, a second tab 256b, a third tab 256c (not shown), and a fourth tab 256d. The tabs 256a, 256b, 256c (not shown), 256d fit into respective tab slots 234a (not shown), 234b, 234c, 234d (not shown) of the recess 234 formed in the housing body first end 200a to rotationally fix the ratchet wheel 256 to the housing body 200. The toothed ratchet wheel 256 has radial teeth formed on one face of the fixed wheel 256 facing the first cover 202.

The sleeve 236 in this embodiment is positioned intermediate to the first cover 202 and the toothed ratchet wheel 256. The sleeve 236 is coupled to a toothed sleeve disk 258 having radial teeth on the surface of the disk 258 facing the toothed ratchet wheel 256. The teeth of the toothed ratchet wheel 256 oppose the teeth of the toothed sleeve disk 258 and rotationally fix the disk 258 to the toothed ratchet wheel 256. The shape of the teeth of the disk 258 (e.g., angle of the tooth surfaces) and the force exerted by the springs 242, 244 determine the frictional force between the toothed sleeve disk 258 and the toothed ratchet wheel 256. That is, the frictional force must be greater than the tension force pulled on the bowstring cocking rope 50 by the bowstring 44. The operation of the bowstring cocking device 224a of FIG. 14 is similar to that of FIGS. 11-13 in that in order to rotate the handle 246 and crank 248, the user must pull the handle and crank away from the first cover 202 so that the sleeve 236 and disk 258 are pulled slightly away from the ratchet wheel 256. In this way, the shaft 214 may be rotated with respect to the housing body 200.

Third Embodiment of Bowstring Cocking Device

FIGS. 15-17 illustrate a third embodiment of a bowstring cocking device 324 having a single shaft 308 and a clutch mechanism 326.

12

Bowstring Cocking Device Structure

Housing Body

Referring to FIG. 15, the bowstring cocking device 324 has a housing body 300 that is generally cylindrical in shape and may be formed from any suitable material (e.g., alloy, stainless steel, ceramic, polymers, etc.). The housing body 300 has a first end 300a that is configured to receive a first cover 302 and a second end 300b that is configured to receive a second cover 304 that are coupled to the housing body using fasteners (e.g., rivets, screws, bolts, etc.) (not numbered). The outer circumference of the first end 300a is generally greater than the outer circumference of the second end 300b. Proximate to the second end of the housing body 300 is a mounting bracket 306 that is configured to attach the bowstring cocking device 324 to the crossbow 10. The housing body also has a first opening 300c proximate the housing body first end 300a. The housing body first opening 300c is generally square shaped and configured to receive a detent 301 having a lever 303 formed at one end.

Shaft, Rope Spools, and Bearings

Running the axial length of the housing body 300 from the first cover 302 to the second cover 304 is a shaft 308 that is rotationally mounted in the housing via a first bearing 314 and a second bearing 316. The shaft has a first end 308a proximate to the first cover 302, a second end 308b proximate to the second cover 304, and a middle portion 308c intermediate to the first and second ends 308a and 308b. The shaft first end 308a is generally square shaped in cross section. The shaft second end 308b and the middle portion 308c are generally circular in cross section. A first spool 310 and a second spool 312 are coupled to the shaft 308 intermediate the shaft first and second ends 308a and 308b. The spools 310, 312 are configured to attach to a first and a second end of the cocking rope 50, respectively, which is configured to be releasably attached to the bowstring 44 by the first and second hooks 50a and 50b. In various embodiments the spools 310, 312 are integrally formed with the shaft 308, and in other embodiments, the spools 310, 312 are coupled to the shaft 308 via suitable fasteners (e.g., screws, pins, weldments, press fit, etc.).

Ratchet Wheel

Positioned intermediate the first spool 310 and the shaft first end 308a is a ratchet wheel 318. The ratchet wheel is received on and rotationally fixed to the shaft 308. The ratchet wheel 318 has a toothed outer circumferential surface 320 and a generally square shaped inner circumference 322. The ratchet wheel 318 is rotationally fixed to the shaft 308 by the interaction of the generally square shaped inner circumference 322 and the square shaped shaft first end 308a.

Clutch Mechanism

The clutch mechanism 326 is received in a generally square shaped recess 300d formed in the housing body first end 300a and is received on the shaft 308 intermediate the ratchet wheel 322 and the first cover 302. The clutch 326 is generally square shaped and may be made from any friction-modifying material Kevlar, metal, alloy, semi-metallic material, sintered metal, resin, carbon material, or woven glass material). The shaft 308 is rotatably received through a circular center hole 328 formed in the clutch mechanism 326. The clutch mechanism 326 is rotationally fixed to the housing body 300 since the shape of the clutch mechanism 326 matches the shape of the recess 300d.

Springs and Washer

A first spring 330 and a second spring 332 are received on the shaft 308 intermediate the clutch mechanism 326 and the first cover 302. A washer 334 is received on the shaft 308 intermediate the first spring 330 and the first cover 302. The

13

first spring 330 is concave towards the washer 334, while the second spring 332 is concave towards the clutch mechanism 326.

Handle and Crank Shaft

As shown in FIG. 15, the bowstring cocking device 32.4 comprises a handle 336 and a crank 338. The handle 336 is rotatably coupled to a first end 338a of the crank 338 using any suitable fastener (e.g., a bolt, a pin, a rivet, weldments, etc.). In various embodiments, the crank 338 may be integrally formed with the handle 336. The handle 336 and the crank 338 are generally perpendicular to each other when attached to one another. The crank 338 also has a second end 338b having a generally square shaped opening 340 that is configured to operatively engage with the shaft squared first end 308a. The shaft 308 is rotationally fixed to the crank 338 by the interaction of the shaft square first end 308a and the crank square shaped opening 340. In various embodiments, the crank 338 may be coupled to the shaft 308 in any suitable manner (e.g., a bolt, a pin, a rivet, a cotter pin, weldments, press fit, etc.).

Safety Mechanism

A screw wheel 342, safety 348 and a detent 303 together form a safety mechanism that allows and prevents the detent 303 from engaging the ratchet wheel teeth 320. The screw wheel 342 has a first portion 344 and a threaded second portion 346. The threaded second portion 346 is threadedly received in a threaded opening 302b formed in the first cover 302. Thus, the screw wheel 342 is both rotatable and axially moveable with respect to the first cover 302. The wheel first portion 344 is positioned intermediate to the crank 338 and the first cover 302 and is, therefore, not received within the housing body 300.

The safety 348 is positioned substantially parallel to the axis of the shaft 308 and has a first end 348a that abuts a surface of the wheel first portion 344. As shown in FIGS. 15 and 17, the safety 348 has a first end 348a, a recessed middle portion 350, and a raised portion 352. The safety 348 is positioned intermediate to the housing body 300 and the screw wheel 342. A spring 301 is positioned intermediate the housing body 300 and a second end 348b of the safety 348 and is configured to bias the safety 348 toward the screw wheel first portion 344. Thus, when the wheel 342 is rotated so that it moves axially with respect to the first cover 302 toward the housing body first cover 302, the safety 348 is biased axially toward the housing second cover 304 thereby aligning the recessed portion 350 with the detent lever second end 303b. If, on the other hand, the screw wheel 342 is rotated in the opposite direction so that the screw wheel second portion moves axially away from the housing body first cover 302, the spring 301 biases the safety 348 toward the first cover 302.

Bowstring Cocking Device Operation

Referring to FIGS. 15, 16 and 17, the safety 348 is moveable between a first position in which the safety raised portion 352 aligns with the detent lever 303 thereby preventing the detent lever second end 303b from engaging the ratchet wheel teeth 320 and a second position in which the safety recessed portion 350 aligns with the detent lever 303 thereby allowing the detent lever second end 303b to engage with the ratchet wheel teeth 320. When the screw wheel 342 is rotated so that the screw wheel first portion 342 moves away from the radial face of the first cover 302, the safety 348 is in the second position so that the safety recessed area 350 aligns with the lever 303. In this position, engagement of the detent lever second end 303b with the ratchet wheel teeth 320 allows the handle 336 and crank 338 to rotate in the counterclockwise direction but prevents rotation of the handle 336 and crank 338 in the clockwise direction. Thus, the user can rotate the

14

handle and crank counterclockwise so that the bowstring cocking rope 50 is wound onto the first and second rope spools 310 and 312 thereby pulling the bowstring 44 into the cocked position.

Once the bowstring 44 is in the cocked position where the trigger device on the crossbow 10 retains the bowstring 44 in the cocked position, the user may rotate the wheel 342 to move the screw wheel first portion 344 axially toward the first cover 302 while pressing on the detent first end 303a so that the safety raised portion 352 aligns with the detent lever 303 thereby preventing the detent lever second end 303b from engaging with the ratchet wheel teeth 320. Additionally, as the screw wheel first portion 344 moves axially toward the first cover 302, the screw wheel second portion 346 abuts and biases the washer 334 axially toward the first and second springs 330 and 332 thereby compressing the springs. As the first and second springs 330 and 332 compress, they exert pressure on the clutch mechanism 326 thereby increasing the frictional forces between the clutch mechanism 326 and the ratchet wheel 318.

Thus, in the first position, friction between the clutch mechanism 326 and the ratchet wheel face 318 prevents the shaft 308 from rotating in either the clockwise or counterclockwise direction. That is, the frictional force between the clutch mechanism 326 and the ratchet wheel front face 318 is larger than the pulling force exerted by the bowstring 44 on the bowstring cocking rope 50. As a result, the clutch mechanism 326 prevents the shaft 308 from rotating in a first direction and an opposite direction until a force is exerted on the handle 336 in combination with the pulling force on the bowstring cocking rope 50 is large enough to overcome the frictional force. Thus, once the bowstring 44 is cocked, the user may turn the handle 336 and crank 338 in the clockwise direction to allow the bowstring 44 to rest against the crossbow trigger mechanism (not shown) in the cocked position. Moreover, further rotation in the clockwise direction allows the user to insert slack in the bowstring cocking rope 50 thereby allowing the user to remove the first and second hooks 50a and 50b from the bowstring 44. Finally, if the user wishes to move the bowstring 44 from a cocked position into the resting position, the user can also continue to rotate the handle 336 and crank 338 in the clockwise direction to move the bowstring into the resting position without having to fire or dry fire the crossbow.

Bowstring Cocking Device Alternate Embodiment

FIGS. 18 and 19 illustrate an embodiment of a bowstring cocking device 424 that is similar to the bowstring cocking device 324 of FIGS. 15-17. For purposes of ease of understanding and clarity, only certain parts will be discussed to highlight the differences in the structure and operation of the embodiment shown in FIGS. 18-19 as compared to the embodiment shown in FIGS. 15-17. The bowstring cocking device 424 contains a housing body 400, a shaft 408 having a first square shaped end 408a and a second end 408b having a first and second rope spool 410 and 412 formed thereon. Similar to the bowstring cocking device 324, first and second bearings 414 and 416 are received on the shaft 408 to allow the shaft to rotate with respect to the housing body 400. A sleeve 400 is rotationally fixed to the shaft first end 308a. A disk 406, a ratchet wheel 412, and first and second springs 430 and 432 are received on, and rotatable with respect to the sleeve 400. A crank 438 is rotationally fixed to the shaft first end 408a and has a handle 436 rotatably coupled thereto by a suitable fastener.

Sleeve

As shown in FIG. 18, the sleeve 404 is rotationally fixed to the shaft 408 since the sleeve 404 contains a square opening

404a that is configured to receive the square shaft first end 408a therein. At the end of the sleeve 404 proximate to the first rope spool 410 is a sleeve disk 404b that is rotationally fixed to the sleeve 404. In various embodiments, the sleeve disk 404b is integrally formed with the sleeve 404. In still other embodiments, the sleeve disk 404b may be formed separately from the sleeve 404 and fastened to the sleeve using any suitable fastener such as pins, rivets, screws, weldments, etc.

Clutch, Springs, and Ratchet Wheel

Encircling the sleeve 404 is the clutch mechanism 406, the ratchet wheel 412, and the first spring 430 and the second spring 432. The clutch mechanism 406 is positioned intermediate the sleeve disk 404b and the ratchet wheel 412. The second spring 432 is positioned intermediate the ratchet wheel 412 and the first spring 430. And the first spring 430 is positioned intermediate the second spring 432 and a first cover 402. The clutch mechanism 406 is generally circular in shape and made from friction-modifying materials (e.g., Kevlar, metal, alloy, semi-metallic material sintered metal, resin, carbon material, or woven glass material). The first spring 430 is concave towards the first cover 402, while the second spring 432 is concave towards the ratchet wheel 412.

Detent

A pin 418 is received through the housing body 400 and engages with a spring 420 at the bottom of the housing body 400. The spring 420 biases the pin 418 radially inward toward the ratchet wheel 412 so that the pin 418 engages the ratchet wheel teeth 412a formed on an outer circumferential surface of the ratchet wheel 412 thereby preventing rotation of the ratchet wheel in the counterclockwise direction while allowing the ratchet wheel to rotate in the clockwise direction. When the pin 418 is pulled out against the bias of the spring 420, the pin 418 no longer engages with the ratchet wheel teeth 412a thereby allowing the ratchet wheel to rotate in both the clockwise and counterclockwise direction.

Bowstring Cocking Device Operation

Referring to FIG. 19, the bowstring cocking device 42.4 is used to pull the bowstring 44 of the crossbow 10 into a cocked position by turning the shaft 408 in the clockwise direction using the handle 436 and crank 438. As the shaft 408 rotates, the ratchet wheel 412 also rotates in the clockwise direction since it is rotationally fixed to the shaft 408 via the clutch mechanism 406 and the sleeve disk 404b. As a result, as the ratchet wheel rotates in the clockwise direction, the pin 418 pops over the ratchet wheel teeth 412a. Once the bowstring is moved into the cocked position, the user may rotate the shaft 408 in the counterclockwise direction by applying rotational force to the shaft 408 via the crank 438 and handle 436. That is, the rotational force applied by the user is sufficient to overcome the frictional force between the ratchet wheel 412, the sleeve disk 404b and the clutch mechanism 406. Thus, the user can place slack in the bowstring cocking rope to allow the first and second hooks 50a and 50b to be removed from the bowstring 44. Once the first and second hooks are removed from the bowstring 44, the user may rotate the shaft 408 in the clockwise direction once again to take up any remaining bowstring cocking rope 50 so that the first and second hooks are positioned adjacent the housing body 400. Once the crossbow is fired, the user may pull the pin 418 out of engagement with the ratchet wheel teeth 412a so that the first and second hooks 50a and 50b may be easily pulled from the housing body 400 and hooked onto the bowstring 44.

If the user wishes to move the bowstring 44 from a cocked position into an un-cocked position, the user may simply apply rotational force to the shaft 408 in the counterclockwise direction so that bowstring cocking rope 50 is wound off the

first and second rope spools 410 and 412. Rotation of the shaft 408 in the counterclockwise direction is controlled by the frictional forces that are exerted between the clutch mechanism 406 and the sleeve disk 404b and the frictional forces that are exerted between the clutch mechanism 406 and the ratchet wheel 412. Thus, if the user releases the handle 436, the shaft will not spin out of control due to the pulling forces exerted on the shaft 408 by the bowstring 44.

Fourth Embodiment of Bowstring Cocking Device

FIGS. 20-24 illustrate a fourth embodiment of a bowstring cocking device 524 that is similar to the bowstring cocking device 24 of FIGS. 3-8. For purpose of ease of understanding and clarity, only certain parts will be discussed to highlight the differences in the structure and operation of the embodiment shown in FIGS. 20-24, as compared to the embodiment shown in FIGS. 3-8.

Bowstring Cocking Device Structure

Referring to FIG. 20, one embodiment of bowstring cocking device 524 is shown. In particular, the bowstring cocking device 524 comprises a handle 500, a crank 502, a housing body 504 having a first cover 506 and second cover 508, a shaft 510, a spool body 520, a ratchet wheel 522, a gear 542, and a clutch mechanism 548.

Housing Body

The first cover 506 and the second cover 508 together enclose the various parts of the bowstring cocking device 524 within the housing body 504. The housing body 504 is generally cylindrical in shape, but may be formed in any suitable shape. In various embodiments, the housing 504 may be formed from any suitable material (e.g., alloy, stainless steel, ceramic, polymers, etc.), and in particular embodiments, the housing 504 is formed from aluminum. The first cover 506 has a first opening 507a and a second opening 507b (as shown in FIGS. 21-24). The inner circumference of the second opening 507b is smaller than the inner circumference of the first opening 507a.

Shaft

The shaft 510 has a first end 510a, a second end 510b and a splined portion 512 intermediate the first and second ends 510a and 510b. The shaft first and second ends 510a, 510b have a generally smooth outer circumference. The shaft second end 510b also has a first and second opening 510c, 510d. The splined shaft 510 is generally cylindrically shaped with axial splines 512a formed on an outer circumference of the shaft intermediate to the first end 510a and the second end 510b where each axial spline is substantially parallel to the axis of the shaft 510.

The shaft 510 is rotatably mounted in the housing 504 so that the shaft first end 510a extends through the first cover second opening 507b and is coupled to a first cap 514 by a first cap screw 514a. In various embodiments, the shaft first end 510a may be integrally formed with the first cap 514, or in other embodiments, the shaft first end 510a may be connected to the first cap 514 using any suitable fastener (e.g., a bolt, a pin, a rivet, weldments, etc.). In addition, the shaft second end 510b extends through the second cover 508 and is coupled to a second cap 515 by a second cap screw 515a. In various embodiments, the shaft second end 510b may be integrally formed with the second cap 515, or in other embodiments, the shaft second end 510b may be connected to the second cap using any suitable fastener (e.g., a bolt, a pin, a rivet, weldments, etc.).

The shaft splines 512a are configured to interact with a splined ring 516, a splined inner circumferential bore 536 of the rope spool body 520, and a splined inner circumferential surface 522a of the ratchet wheel 522. The second end 510b of the shaft 510 is configured to receive a disk 518, the splined

ring 516, the clutch mechanism 548, a washer 523, the first spring 526, and the second spring 528.

Rope Spool Body

The rope spool body 520 comprises a first end 520a and a second end 520b. The rope spool body 520 is generally cylindrical in shape and has a first rope spool 532 and a second rope spool 534. The rope spools 532, 534 provide a space for the bowstring cocking rope 50 to wind around when the bowstring 44 is pulled from the resting position into the cocked position. The rope spool body 520 is received on, axially moveable with respect to, and rotationally fixed to the shaft 510 by the splined bore 536. The outer circumference of the rope spool body first end 520a is generally circular with a hexagonally shaped end 521. In addition, the rope spools 532 and 534 are configured to operatively attach to a first and a second end of the bowstring cocking rope 50. Referring to FIG. 21, a first opening 524 and a second opening 526 in the housing body 504 are positioned above the first rope spool 532 and the second rope spool 534, respectively, and allow the cocking rope 50 (not shown) to enter the housing body 504 and wind around the rope spools 532, 534.

Gear

Referring again to FIG. 20, the gear 542 has a generally hexagonal inner circumference 544 that is configured to operatively engage with the rope spool body hexagonally shaped end 521 to rotationally fix the rope spool body 520 to the gear 542. That is, the gear inner circumference 544 and rope spool body hexagonally shaped end 521 are both substantially hexagonal in shape where the gear inner circumference 544 is slightly larger than the rope spool body hexagonally shaped end 521. In various embodiments, the gear 542 may be integrally formed with the rope spool body 520. In still other embodiments, the gear 542 may be rotationally fixed to the shaft 510 via a splined engagement similar to the splined engagement between the rope spool body 520 and the shaft splines 512a.

Ratchet Wheel and Detent

The ratchet wheel 522 is positioned intermediate the first cover 506 and the gear 542. The ratchet wheel 522 has (1) the splined inner circumference surface 522a and is received on the shaft splined portion 512, and (2) a toothed outer circumferential surface 522b. The ratchet wheel 522 is positioned on the shaft 510 proximate to the shaft first end 510a. A detent 546 is operatively received in the housing 504 such that one end of the detent 546 may be moved into and out of engagement with the ratchet wheel toothed outer circumferential surface 522b to prevent unwanted rotation of the ratchet wheel 522 in one direction.

A pin 545 is received through the detent 546 and is received in the housing body 504. The pin 545 allows the detent to pivot into an out of engagement with the ratchet wheel teeth 522b. A spring 547 mounted intermediate the housing body and the detent 546 biases the detent 546 radially inward toward the ratchet wheel 522 so that the detent 546 engages the ratchet wheel teeth 522a thereby preventing rotation of the ratchet wheel in the counterclockwise direction while allowing the ratchet wheel 522 to rotate in the clockwise direction. When the detent 546 is rotated outward, the detent 546 no longer engages with the ratchet wheel teeth 522a thereby allowing the ratchet wheel to rotate in both the clockwise and counterclockwise direction.

Clutch Mechanism

The clutch mechanism 548 is received on the shaft 510 adjacent the shaft second end 510b. The clutch 548 is also received in, and rotationally fixed to, the second cover 508. That is, the second cover 508 contains a substantially square recess that receives the substantially square clutch mechanism

548 so that the clutch mechanism is rotationally fixed to the second cover 508 and the housing 504. It should be understood that the clutch mechanism 548 may be rotationally fixed to the second cover 508 by other suitable means. The clutch 548 has a smooth, circular inner circumferential surface 550 for receiving the shaft second end 510b. In various embodiments, the clutch 548 is generally square shaped and made from friction-modifying materials (e.g., Kevlar, metal, alloy, semi-metallic material, sintered metal, resin, carbon material, or woven glass material).

Crank Shaft, Gear and Handle

Still referring to FIG. 20, the handle 500 is coupled to a first end 502a of the crank 502. In various embodiments, the crank 502 may be connected to the handle 500 using any suitable fastener (e.g., a bolt, a pin, a rivet, weldments, etc.). The crank 502 and the handle 500 are generally perpendicular to each other when attached. The crank 502 also has a second end 502b with an opening 502c that is configured to operatively engage with a gear crank shaft 511. The gear crank shaft 511 is rotationally fixed to the crank 502 since the shape of the crank opening 502c matches the shape of an end 511a of the gear crank shaft 511. That is, the crank opening 502c and the end 511a of the gear crank shaft 511 are both substantially square in shape. In various embodiments, the crank 502 may be coupled to the gear crank shaft 511 in any suitable manner (e.g., a bolt, a pin, a rivet, a cotter pin, weldments, etc.).

The gear crank shaft 511 is operatively coupled to a crank gear 513 so that the gear crank shaft is rotationally fixed to the crank gear 513. In various embodiments, the crank gear 513 is integrally formed with the gear crank shaft 511. In various other embodiments, the crank gear 513 may be connected to the gear crank shaft 511 using any suitable fastener (e.g., a bolt, a pin, a rivet, a cotter pin, weldments, etc.). The crank gear 513 has teeth that match the teeth of the gear 542.

Parts Enclosed Inside the Housing Body

Referring to FIG. 21, when the bowstring cocking device is assembled, the following parts fit inside the housing body 504 beginning at the first cover 506 and progressing along to the second cover 508: the ratchet wheel 522, the gear 542, the rope spool body 520, the disk 518, the splined ring 516, the clutch mechanism 548, the washer 523, the first spring 526, and the second spring 528 and running through the length of the inner circumference of each of these parts is the shaft 510. Bowstring Cocking Device Operation

First Position

Referring to FIGS. 21 and 23, the bowstring cocking device 524 is shown in a first position where the shaft 510 is moved axially into a first position with respect to the housing body 504 such that the splined portion 512 is engaged with the ratchet wheel splined inner circumferential surface 522a and is disengaged from the splined ring 516. In the first position, the splined ring 516 is positioned on the smooth surface of the shaft second end 510b. The bowstring cocking device 524 may be moved into the first position as shown in FIGS. 21 and 23 by pushing the second cap 515 axially toward to the first cover 506. Because the second cap 515 is coupled to the shaft 510, moving the second cap 515 axially toward the first cover 506 causes the shaft 510 to move axially (toward the right with regard to FIG. 21) so that the splined portion 512 engages with the inner circumferential splined surface 522a of the ratchet wheel 522, which causes the ratchet wheel 522 to be rotationally fixed with the shaft 510 while the splined ring 516 is allowed to rotate with respect to the shaft 510.

Referring again to FIG. 20, the teeth of the crank gear 513 engage with the teeth of the gear 542 so that rotation of the crank gear 513 in the clockwise direction causes the gear 542 to rotate in a counterclockwise direction (with respect to the

view of FIG. 23). Thus when the crank 502 is turned clockwise, the gear crank shaft 511 also rotates clockwise in turn causing the crank gear 513 to rotate clockwise. This, in turn, rotates the gear 542 in the opposite, counter-clockwise, direction. Because the gear 542 is rotationally fixed to the shaft 510 via the spool body 520, and the ratchet wheel 522 and the spool body 520 are rotationally fixed to the shaft 510, rotation of crank 502 and crank shaft 511 clockwise causes the spool body 520 to rotate counterclockwise. As a result, as the rope spool body 520 rotates counterclockwise, it winds up the bowstring cocking rope 50 and pulls the bowstring 44 into the cocked position. Because the spring 547 biases the detent 546 into the ratchet wheel toothed outer circumferential surface 522b, the detent 546 prevents unwanted rotation of the ratchet wheel 522 in the clockwise direction. As a result, the spool body 520 is also prevented from rotating in the clockwise direction since the ratchet wheel 522 and the spool body 520 is rotationally fixed to the shaft 510. In this way, the spool body 520 will not inadvertently rotate clockwise in response to the bias exerted on the bowstring cocking rope 50 by the bowstring 44.

Second Position

FIGS. 22 and 24 show the shaft 510 in a second position where the shaft splined portion 512 is disengaged from the ratchet wheel 522 and is engaged with the splined ring 516. In the second position, either the bowstring cocking rope 50 may be slightly released so that the first and second hooks 50a and 50b may be removed from the bowstring 44 once the bowstring is in the cocked portion, or (2) the bowstring 44 may be moved from the firing position into the resting position without firing or dry firing the crossbow 10. The bowstring cocking device is moved into the second position when the user pushes the first cap 514 axially away from the first cover 506, causing the shaft 510 to move axially rearward (e.g., to the left in FIG. 22) so that the shaft splined portion 512 engages the splined ring 516 and disengages from the splined ratchet wheel 522. Because the inner circumference of the first cover second hole 507a is larger than the outer circumference of the shaft first end 510a but smaller than the outer circumference of the first cap 514, the shaft moves axially through the second hold 507a and the first cap 514 functions as a stop to prevent the user from pushing the first cap 514 through the first cover 506.

Thus, when the bowstring cocking device 524 is in the second position, the ratchet wheel 522 no longer prevents the shaft 510 from rotating in the clockwise direction since the ratchet wheel 522 is no longer rotationally fixed to the shaft 510. Instead, the engagement of a face of the splined ring 516 with a face of the clutch mechanism 548 prevents the shaft 510 from spinning freely. That is, the frictional force between the face of the splined ring 516 and the corresponding face of the clutch mechanism 548 is greater than the pulling force exerted on the bowstring cocking rope 50 by the bowstring 44. As a result, the shaft remains rotationally fixed to the housing through the clutch mechanism 548 until the user exerts sufficient force on the shaft to overcome the frictional force by turning the handle 500 and the crank 502 in the counterclockwise direction, which in turn causes the shaft 510 to rotate in the clockwise direction, thereby letting the bowstring cocking rope 50 out from the rope spools 532, 534. In this way, the user can either release tension on the bowstring cocking rope to allow the user to remove the first and second hooks 50a and 50b (not shown) from the bowstring 44 or to release the bowstring 44 from the cocked position into the resting position in a controlled manner without firing or dry firing the crossbow.

Fourth Embodiment of Bowstring Cocking Device

FIGS. 25-27 illustrate an embodiment of a bowstring cocking device 624 that is similar to the bowstring cocking device 424 of FIGS. 18-19. For purposes of ease of understanding and clarity, only certain parts will be discussed to highlight the differences in the structure and operation of the embodiment shown in FIGS. 25-27 as compared to the embodiment shown in FIGS. 18-19. Referring particularly to FIG. 25, the bowstring cocking device 624 contains a first housing body 600, a second housing body 602, a first rope spool body 604 having a first substantially square shaped end 604a and a first rope spool 606. The bowstring cocking device 624 also contains a second rope spool body 608 having (1) a first generally rectangular shaped end 608a, (2) a second generally square shaped end 608b, and (3) a second rope spool 610 formed thereon intermediate the shaft first and second ends 608a, 608b. Together, the first and second rope spool bodies form a shaft 607 that comprises the first rope spool body 604 and the shaft first and second ends 608a and 608b. Similar to the bowstring cocking device 424, a first and a second bearing 612 and 616 are respectively received on the first and second rope spool bodies 604, 608 to allow the rope spool bodies 604, 608 to rotate with respect to the housing bodies 600, 602, respectively. A sleeve 614 attached to a ring 618 are rotationally fixed to the second rope spool body first end 608a, as described below. A clutch mechanism 620, a ratchet wheel 622, a first spring 624 and a second spring 626 are received on, and is rotatable with respect to the sleeve 614 and sleeve ring 618. A crank 628 (1) is rotationally fixed to the first rope spool body first end 604a and (2) has a handle 630 rotatably coupled thereto by a suitable fastener.

Sleeve

Still referring to FIG. 25, the sleeve 614 and sleeve ring 618 are rotationally fixed to the second rope spool body 608 since the sleeve 618 contains a generally square opening 632 that is configured to receive the generally square shaped second rope spool body first end 608a therein. At the end of the sleeve 614 proximate to the first rope spool 606 is the sleeve disk 618 that is rotationally fixed to the sleeve 614. In various embodiments, the sleeve disk 618 is integrally formed with the sleeve 614. In still other embodiments, the sleeve disk 618 may be formed separately from the sleeve 614 and fastened to the sleeve 614 using any suitable fastener such as pins, rivets, screws, weldments, etc.

Clutch, Springs, and Ratchet Wheel

Encircling the sleeve 618 is the clutch mechanism 620, the ratchet wheel 622, the first spring 624 and the second spring 626. The ratchet wheel 622 has a toothed outer circumference 622a. The clutch mechanism 620 is positioned intermediate the sleeve disk 618 and the ratchet wheel 622. The ratchet wheel 622 is positioned intermediate the sleeve disk 618 and the first spring 624. The first spring 624 is positioned intermediate the ratchet wheel 622 and the second spring 626. And the second spring 626 is positioned intermediate the first spring 624 and the second rope spool 610. The clutch mechanism 620 is generally circular in shape and made from friction-modifying materials Kevlar, metal, alloy, semi-metallic material, sintered metal, resin, carbon material, or woven glass material). The first spring 624 is concave towards the second spring 626, while the second spring 626 is concave towards the first spring 624.

Detent

Referring to FIG. 26, a detent 636 is received through the second housing body 602 and engages with a spring 638 at the bottom of the second housing body 602. The spring 638 biases a first end 636a of the detent 636 radially outward which in turn causes the second end 636b of the detent 636 to

move radially inward toward the ratchet wheel **622** so that the detent second end **636b** engages the ratchet wheel teeth **622a** thereby preventing rotation of the ratchet wheel **622** in the counterclockwise direction while allowing the ratchet wheel **622** to rotate in the clockwise direction. When the detent first end **636a** is pushed radially inward against the bias of the spring **638**, the detent second end **636b** moves radially outward so that it no longer engages with the ratchet wheel teeth **622a** thereby allowing the ratchet wheel **62.2** to rotate in both the clockwise and counterclockwise direction.

Bowstring Cocking Device Operation

Referring to FIG. 27, the bowstring cocking device **624** is used to pull the bowstring **44** of the crossbow **10** into a cocked position by turning the shaft **607** in the clockwise direction using the handle **630** and crank **628**. As the shaft **607** rotates, the ratchet wheel **622** also rotates in the clockwise direction since it is rotationally fixed to the shaft **607** via the clutch mechanism **620** and the sleeve disk **618**. As a result, as the ratchet wheel **622** rotates in the clockwise direction, the detent second end **638b** pops over the ratchet wheel teeth **622a**. Once the bowstring is moved into the cocked position, the user may rotate the shaft **607** in the counterclockwise direction by applying rotational force to the shaft **607** via the crank **628** and handle **630**. That is, the rotational force applied by the user is sufficient to overcome the frictional force between the ratchet wheel **622**, the sleeve disk **618** and the clutch mechanism **620**. Thus, the user can place slack in the bowstring cocking rope **50** to allow the first and second hooks **50a** and **50b** to be removed from the bowstring **44**. Once the first and second hooks are removed from the bowstring **44**, the user may rotate the shaft **607** in the clockwise direction once again to take up any remaining bowstring cocking rope **50** so that the first and second hooks **50a** and **50b** are positioned adjacent the housing bodies **600**, **602**. Once the crossbow **10** is fired, the user may push the detent first end **636a** so that the detent second end **636b** moves out of engagement with the ratchet wheel teeth **622a** so that the first and second hooks **50a** and **50b** may be easily pulled from the housing bodies **600**, **602** and hooked onto the bowstring **44**.

If the user wishes to move the bowstring **44** from a cocked position into an un-cocked position, the user may simply apply rotational force to the shaft **607** in the counterclockwise direction so that bowstring cocking rope **50** is wound off the first and second rope spools **606** and **610**. Rotation of the shaft **607** in the counterclockwise direction is controlled by the frictional forces that are exerted between the clutch mechanism **620** and the sleeve disk **618** and the frictional forces that are exerted between the clutch mechanism **620** and the ratchet wheel **622**. Thus, if the user releases the handle **630**, the shaft **607** will not spin out of control due to the pulling forces exerted on the shaft **607** by the bowstring **44**.

It should be understood to one of skill in the art that by placing the ratchet wheel, the clutch mechanism and the sleeve disk intermediate the first and second spools, the overall size of the bowstring cocking device can be reduced.

Bowstring Cocking Device Alternate Embodiment

Referring to FIGS. 28-30 an alternative embodiment of a bowstring cocking device **724** is shown having a dual shaft design as opposed to the single shaft design shown in the bowstring cocking device **624** embodiment of FIGS. 25-27. Thus, for purposes of ease of understanding and clarity, only certain parts will be discussed to highlight the differences in the structure and operation of the embodiment shown in FIGS. 28-30 as compared to the embodiment shown in FIGS. 25-27.

Handle and Crank

The handle **700** and crank shaft **702** are coupled to a gear shaft **704**. That is, the crank second end **702b** contains a square opening **702c** that mates with, and is rotationally fixed to, a first end **704a** of the shaft **704**. Additionally, this alternate embodiment utilizes a bolt that runs the length of the second rope spool body **708** that is used to secure the second rope spool body to the first rope spool body.

Housing Bodies and Rope Spool Bodies

The bowstring cocking device **724** contains a first housing body **710**, a second housing body **712**, a first rope spool body **714** having a first substantially square shaped end **714a**, a second substantially square shaped end **714b**, and a first rope spool **716**. The bowstring cocking device **724** also contains the second rope spool body **708** having a first end **708a** and a second end **708b**, and having a second rope spool **718** formed thereon intermediate the second rope spool body first and second ends **708a**, **708b**. The second rope spool body **708** has a generally square shaped inner circumference **708c**. The first rope spool body second end **714b** is received by the second rope spool body inner circumference **708c** so that the first and second rope spool bodies are rotationally fixed to one another. Also received on the first rope spool body second end **714b** is a clutch mechanism **720**, a sleeve **722**, a ratchet wheel **725**, a first spring **726**, and a second spring **728**. When the first rope spool body second end **714b** is received in the second rope spool body inner circumference **708c**, the first and second rope spool bodies together define a shaft **730**. Similar to the bowstring cocking device **624**, first and second bearings **732**, **734** are received on the first and second rope spool bodies **714**, **708** to allow the rope spool bodies **714**, **708** to rotate with respect to the housing bodies **710**, **712**.

Sleeve

As shown in FIG. 25, the sleeve **722** is rotationally fixed to the first rope spool body **714** since the sleeve **722** contains a generally square opening **722a** that is configured to receive the square first rope spool body first end **714a** therein. At the end of the sleeve **722** intermediate the clutch mechanism **720** and the ratchet wheel **725** is a sleeve disk **736** that is rotationally fixed to the sleeve **722**. The ratchet wheel **725**, and the first and second springs **726**, **728** are received on, and rotatable with respect to the sleeve **722**. In various embodiments, the sleeve disk **736** is integrally formed with the sleeve **722**. In still other embodiments, the sleeve disk **736** may be formed separately from the sleeve **722** and fastened to the sleeve **722** using any suitable fastener such as pins, rivets, screws, weldments, etc.

Clutch, Springs, and Ratchet Wheel

Encircling the first spool body second end **714b** is the sleeve disk **736** and the sleeve **722**. The clutch **720** and the ratchet wheel **725** are received on the sleeve **722**. That is, the clutch **720** is positioned intermediate the sleeve disk **736** and the ratchet wheel. Also encircling the sleeve **722** is the first spring **726** and the second spring **728**, which are positioned intermediate the second rope spool body **708** and the ratchet wheel **725**. The ratchet wheel **725** has a toothed outer circumference **725a**. The clutch mechanism **720** is generally circular in shape and made from friction-modifying materials (e.g., Kevlar, metal, alloy, semi-metallic material, sintered metal, resin, carbon material, or woven glass material). The first spring **726** is concave towards the second spring **728**, while the second spring **728** is concave towards the first spring **726**.

Gear, Crank Gear, and Crank Gear Shaft

Still referring to FIG. 28, the crank second end opening **702c** is configured to operatively engage with a gear crank shaft **704**. The gear crank shaft **704** is rotationally fixed to the crank **702** since the shape of the crank opening **702c** matches

23

the shape of an end **704a** of the gear crank shaft **704**. That is, the crank opening **702c** and the gear crank shaft end **704a** are both substantially square in shape. In various embodiments, the crank **702** may be coupled to the gear crank shaft **704** in any suitable manner (e.g., a bolt, a pin, a rivet, a cotter pin, weldments, etc.).

The gear crank shaft **704** is operatively coupled to a crank gear **705** so that the gear crank shaft **704** is rotationally fixed to the crank gear **705**. In various embodiments, the crank gear **705** is integrally formed with the gear crank shaft **704**. In various other embodiments, the crank gear **705** may be connected to the gear crank shaft **704** using any suitable fastener (e.g., a bolt, a pin, a rivet, a cotter pin, weldments, etc.). The crank gear **705** has teeth that match the teeth of a gear **738**. The gear **738** has a substantially square inner circumference to receive the first rope spool body first end **714a** so that the first rope spool body **714** is rotationally fixed to the gear **738**.
Bowstring Cocking Device Operation

Referring to FIG. 29, the bowstring cocking device **724** is used to pull the bowstring **44** of the crossbow **10** (FIG. 1) into a cocked position by turning the crank shaft **702** in the clockwise direction using the handle **700**. Referring again to FIG. 28, the teeth of the crank gear **705** engage with the teeth of the gear **738** so that rotation of the crank gear **705** in the clockwise direction (with respect to FIG. 29) causes the gear **738** to rotate in a counterclockwise direction. Thus when the crank **702** is turned clockwise, the gear crank shaft **704** also rotates clockwise in turn causing the crank gear **705** to rotate clockwise. This, in turn, rotates the gear **738** in the opposite, counter-clockwise, direction. Because the gear **738** is rotationally fixed to the first rope spool body **714** and the second rope spool body **708** is rotationally fixed to the first rope spool body **714**, rotation of the crank **702** and the gear crank shaft **704** clockwise causes the first and second spool bodies **714**, **708** to rotate counterclockwise. As a result, as the rope spool bodies **714**, **708** rotate counterclockwise, they wind up the bowstring cocking rope **50** and pull the bowstring **44** into the cocked position. The overall operation of the bowstring cocking device **724** is substantially similar to the bowstring cocking device **624** and a detailed description is omitted for brevity.

Conclusion

In all of the various embodiments described above, various clutch mechanism are used to control the rotation of a shaft in a bowstring cocking device. As such, frictional forces between a disk that is rotationally fixed to the shaft and a clutch mechanism that is either (1) rotationally fixed to a housing body or (2) positioned intermediate to a ratchet wheel and a disk rotationally fixed to the shaft help to control the rotation of the shaft when force is exerted on the shaft by the bowstring. The various configurations also allow the user to either (1) provide slack in the bowstring cocking rope so that the user can remove the hooks connecting the bowstring cocking rope to the bowstring, or (2) move the bowstring from a cocked position into an un-cocked position without the user firing or dry firing the crossbow.

What is claimed:

1. A crossbow drawing mechanism for use with a crossbow, the crossbow drawing mechanism comprising:

- a. a housing having a cavity formed therein;
- b. a first shaft positioned at least partially in the housing cavity and rotatable with respect to the housing, the first shaft comprising:
 - i. a first end, and
 - ii. a second end,
- c. a first spool rotatably fixed to the first shaft proximate the first shaft first end and positioned in the housing cavity;

24

- d. a second spool rotatably fixed to the first shaft intermediate the first spool and the first shaft second end and positioned in the housing cavity;
- e. a ratchet wheel received on the first shaft intermediate the first spool and the second spool, wherein the ratchet wheel is rotatable with respect to the first shaft and the ratchet wheel comprising a plurality of teeth that are adapted to receive a pawl for rotationally fixing the ratchet wheel to the housing; and
- f. a crank operatively coupled to the first shaft such that when the crank is rotated the shaft rotates, wherein the ratchet wheel rotationally fixes the first shaft to the housing in a first direction and allows the shaft to rotate with respect to the housing in an opposite second direction.

2. The crossbow drawing mechanism of claim 1, further comprising a clutch positioned intermediate the ratchet wheel and one of the first spool and the second spool, wherein:

the clutch rotationally fixes the ratchet wheel to the first shaft.

3. The crossbow drawing mechanism of claim 2, wherein the clutch further comprises:

- a. a sleeve that is rotationally fixed to the first shaft;
- b. a clutch disc positioned intermediate the ratchet wheel and the sleeve; and
- c. at least one spring positioned intermediate the second spool and the ratchet wheel, wherein the clutch disc is compressed between the ratchet wheel and the sleeve.

4. The crossbow drawing mechanism of claim 1, further comprising a pawl rotatably coupled to the housing so that a portion of the pawl is releasably engaged with the teeth formed on a periphery of the ratchet wheel.

5. The crossbow drawing mechanism of claim 1, further comprising a second shaft at least partially received in the housing, wherein

- a. the second shaft is operatively coupled to the first shaft;
- b. the crank is coupled to the second shaft;
- c. rotation of the crank in the first direction causes the first shaft to rotate in the second direction and rotation of the crank in the second direction causes the first shaft to rotate in the first direction.

6. The crossbow drawing mechanism of claim 1, wherein the crank is directly coupled to the first shaft first end.

7. A crossbow drawing mechanism for use with a crossbow, the crossbow drawing mechanism comprising:

- a. a generally cylindrical housing;
- b. a shaft positioned at least partially within the generally cylindrical housing and rotatable with respect to the housing, the shaft comprising:
 - i. a first end, and
 - ii. a second end,
- c. a first spool rotatably fixed to the shaft;
- d. a second spool rotatably fixed to the shaft and spaced apart from the first spool;
- e. a ratchet wheel received on the shaft intermediate the first spool and the second spool, wherein the ratchet wheel is rotatable with respect to the shaft and comprises a plurality of teeth that are adapted to receive a pawl for rotationally fixing the ratchet wheel to the housing; and
- f. a clutch positioned intermediate the first spool and the second spool such that the clutch rotationally fixes the ratchet wheel to that shaft;

wherein

 - i. when the shaft is rotated in a first direction, the ratchet wheel prevents the shaft from rotating with respect to the housing, and

25

- ii. when the shaft is rotated in an opposite second direction, the ratchet wheel rotates in the second direction with the shaft.

8. The crossbow drawing mechanism of claim 7, wherein when a sufficient force is exerted on the shaft in the first direction, the shaft rotates with respect to the housing in the first direction. 5

9. The crossbow drawing mechanism of claim 7, further comprising a handle coupled to the first end of the shaft.

10. The crossbow drawing mechanism of claim 7, further comprising a pawl coupled to the housing, wherein a portion of the pawl engages with the plurality of teeth formed on the outer periphery of the ratchet wheel to rotationally lock the ratchet wheel to the housing to prevent the shaft from rotating in the first direction while allowing the shaft to rotate in the second direction. 15

11. The crossbow drawing mechanism of claim 7, wherein the clutch further comprises a clutch disk positioned intermediate the ratchet wheel and one of the first spool and the second spool, wherein the clutch disk rotationally fixes the ratchet wheel to the one of the first spool or the second spool. 20

12. The crossbow drawing mechanism of claim 7, wherein the clutch further comprises:

- a. a sleeve received on the shaft intermediate the ratchet wheel and one of the first spool or the second spool;
- b. a clutch disk positioned intermediate the sleeve and the ratchet wheel; and
- c. at least one spring positioned intermediate the one of the first spool or the second spool and the ratchet wheel, wherein 25
 - i. the sleeve is rotationally fixed to the shaft, and
 - ii. the clutch disk rotationally fixes the ratchet wheel to the shaft via the sleeve.

13. The crossbow drawing mechanism of claim 12, wherein one of the ratchet wheel or the sleeve is axially moveable with respect to the shaft. 35

14. The crossbow drawing mechanism of claim 7, further comprising:

- a. a second shaft rotatably mounted in the housing, the second shaft having a first end and an opposite second end; 40
- b. a handle coupled to the second shaft first end; wherein the second shaft is operatively coupled to the shaft, when the handle is rotated in the first direction, the second shaft rotates in the first direction and the shaft rotates in the second direction, and 45 when a sufficient force is exerted on the handle in the second direction, the second shaft rotates in the second direction and the shaft rotates in the first direction with respect to the ratchet wheel. 50

15. A crossbow and a crossbow drawing mechanism comprising:

- a. an elongated body;
- b. a first limb coupled to a first end of the elongated body;
- c. a second limb coupled to the elongated body first end;
- d. a bowstring having a bowstring first end operatively coupled to the first limb and a bowstring second end operatively coupled to the second limb;
- e. a crossbow drawing mechanism comprising: 60
 - i. a housing coupled to the elongated body distal from the first end,
 - ii. a first shaft having a first end and a second end, wherein at least a portion of the first shaft is rotatably mounted in the housing,

26

- iii. a first spool mounted on the first shaft proximate the first end of the first shaft and rotatably fixed to the first shaft,

- iv. a second spool mounted on the first shaft intermediate the first spool and the second end of the first shaft and rotatably fixed to the first shaft,

- v. a ratchet wheel rotatably mounted on the first shaft intermediate the first spool and the second spool, the ratchet wheel comprising a plurality of teeth that are adapted to receive a pawl for rotationally fixing the ratchet wheel to the housing; and

- vi. a clutch mounted on the first shaft intermediate one of the first spool or the second spool and the ratchet wheel,

wherein the clutch and the ratchet wheel rotationally fix the shaft to the housing in a first direction and allow the shaft to rotate with respect to the housing in an opposite second direction.

16. The crossbow and the crossbow drawing mechanism of claim 15, further comprising a pawl pivotally mounted to the housing such that a portion of the pawl is operatively engaged with the plurality of teeth formed on a periphery of the ratchet wheel.

17. The crossbow and the crossbow drawing mechanism of claim 16, further comprising a second shaft rotatably mounted in the housing and operatively coupled to the first shaft by a gear, wherein

- a. the crank is coupled to the second shaft,
- b. when the crank is rotated in the first direction, the second shaft rotates in the first direction and the first shaft rotates in the second direction, and
- c. when the pawl is rotated with respect to the housing, the pawl disengages from the ratchet wheel so that when the crank is rotated in the second direction, the second shaft rotates in the second direction and the first shaft rotates in the first direction.

18. The crossbow and the crossbow drawing mechanism of claim 15, the clutch further comprising:

- a. a sleeve rotatably fixed to the first shaft and positioned intermediate the ratchet wheel and the one of the first spool or the second spool; and
- b. a clutch disc rotatably mounted on the first shaft intermediate the sleeve and the ratchet wheel, wherein the clutch disk rotationally fixes the ratchet wheel to the first shaft via the sleeve.

19. The crossbow and the crossbow drawing mechanism of claim 15, further comprising a crank operatively coupled to the first shaft.

20. The crossbow and the crossbow drawing mechanism of claim 18, further comprising a second shaft rotatably mounted in the housing and operatively coupled to the first shaft by a gear, wherein

- a. a crank is coupled to the second shaft,
- b. when the crank is rotated in the first direction, the second shaft rotates in the first direction and the first shaft rotates in the second direction, and
- c. when a sufficient force is applied to the crank in the second direction, the second shaft rotates in the second direction and the first shaft rotates in the first direction against the frictional force exerted between a sleeve and the ratchet wheel.